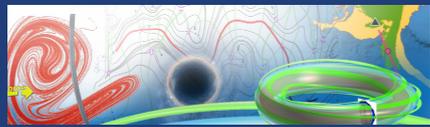


# Uncovering how conservative backbone curves survive the advent of forcing and damping



Mattia Cenedese

Ph.D. Student, Chair in Nonlinear Dynamics

Joint work with Prof. Dr. George Haller

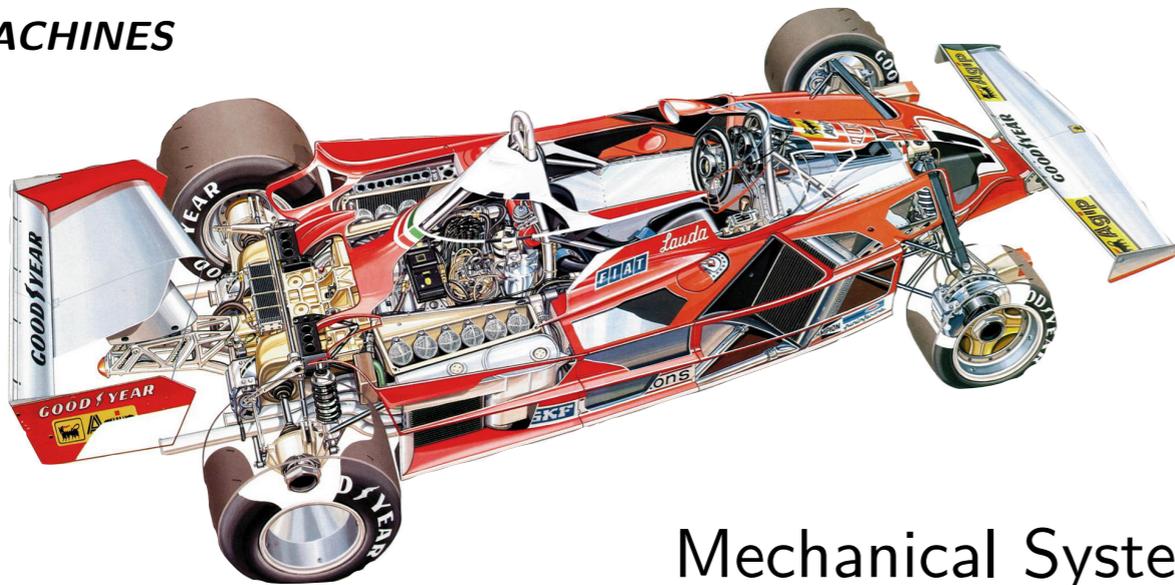
**ETH** zürich



# The Role of Periodic Orbits in Dynamics

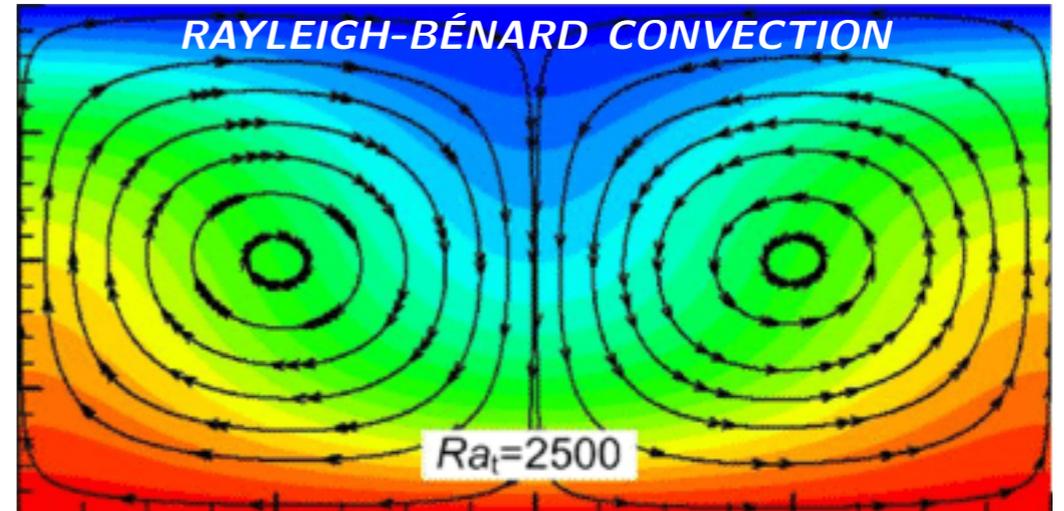
**Periodic orbit:** *motion that repeats identically after a finite period of time*

MACHINES



Mechanical Systems

RAYLEIGH-BÉNARD CONVECTION

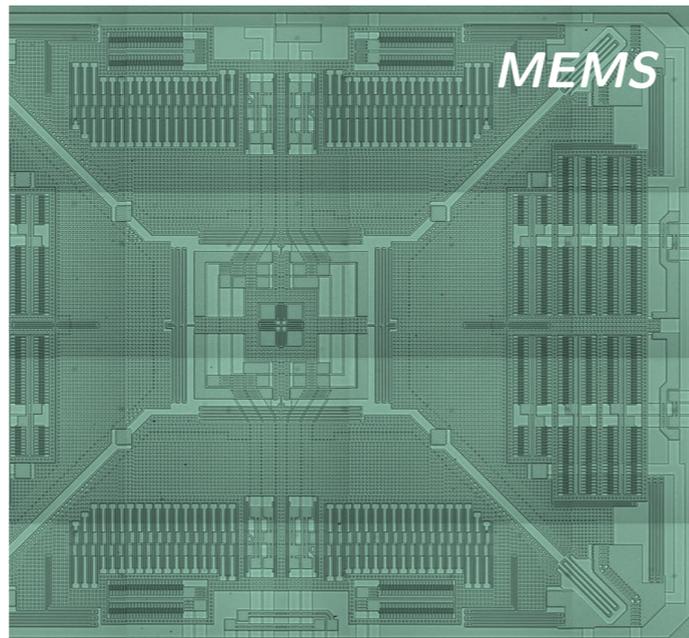


Fluid Dynamics

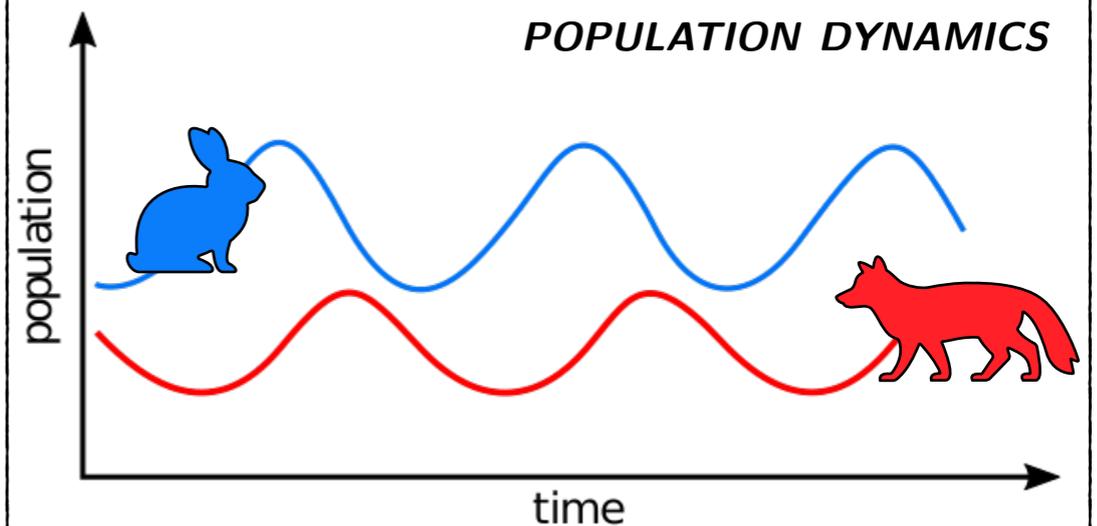
STRUCTURES



MEMS



POPULATION DYNAMICS



Socio-biological Dynamics

# The Role of Periodic Orbits in Dynamics

**Periodic orbit:** *motion that repeats identically after a finite period of time*

**MACHINES**

**RAYLEIGH-BÉNARD CONVECTION**

$Ra_1=2500$

**Fluid Dynamics**

**Mechanical Systems**

*conservative*      *time-periodic*

$\dot{x} = f(x) + \varepsilon g(x, t, \varepsilon),$

$0 < \varepsilon \ll 1$

**STRUCTURES**

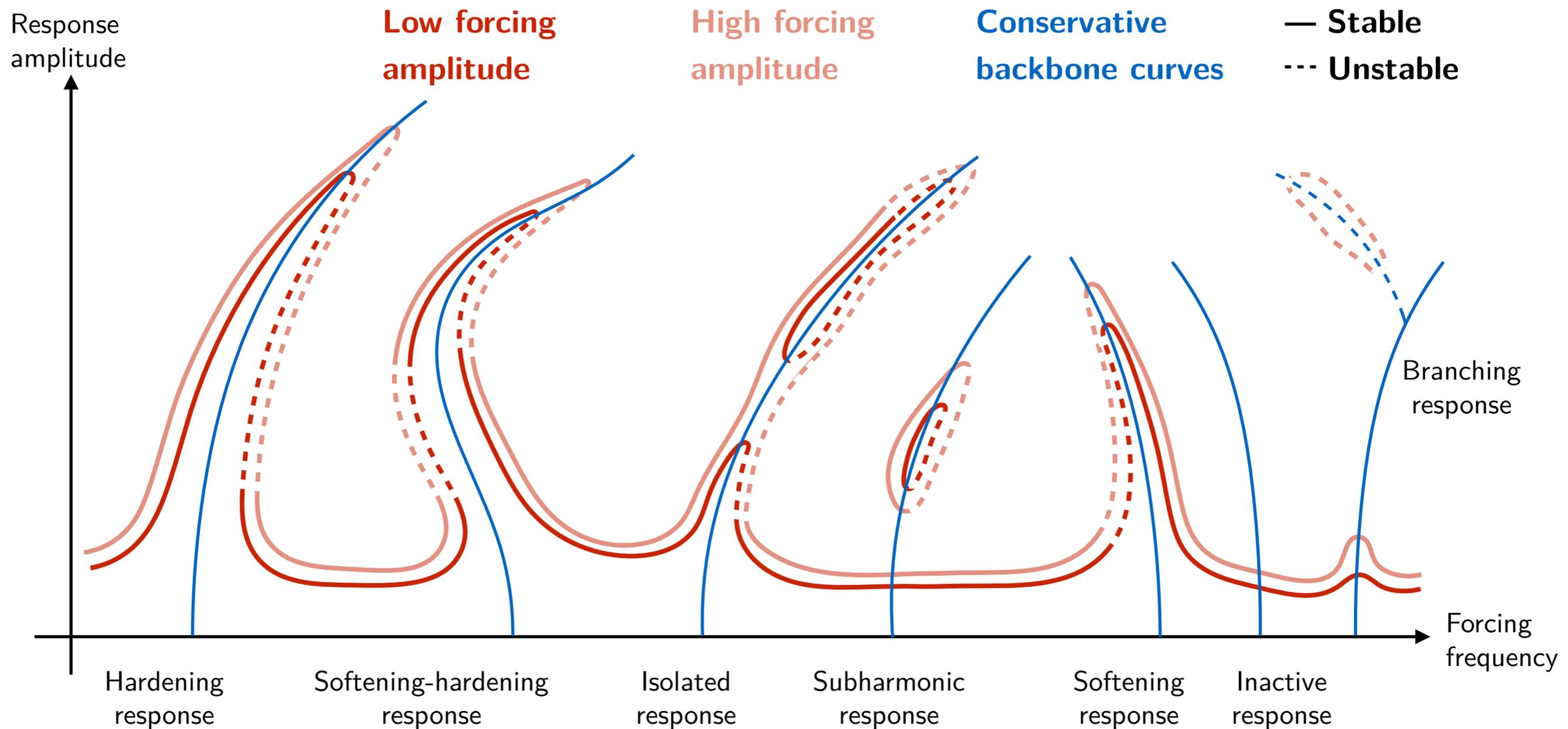
**POPULATION DYNAMICS**

**Socio-biological Dynamics**

*Can we predict existence and stability of periodic orbits of the perturbed system starting from those of the conservative system?*

# Motivations: the case of Mechanical Vibrations

Consider  $N$  coupled, periodically forced and damped oscillators for **arbitrary motion amplitude**. Some nonlinear phenomena



# Motivations: the case of Mechanical Vibrations

*Why would practitioners capitalize on analytical tools?*

*Computational speed-up for studies of the effect forcing & damping terms*

*Find isolas: identification is challenging from numerical continuation*

*Validate and extend experimental routines using the phase-lag quadrature*

**Available methods:**

- *Asymptotic expansions from an equilibrium<sup>1</sup>*
- *LSM & SSM<sup>2</sup>*
- *Energy-type arguments<sup>3</sup>*
- *Melnikov methods*

Hardening

Softening-hardening

Isolated

Subharmonic

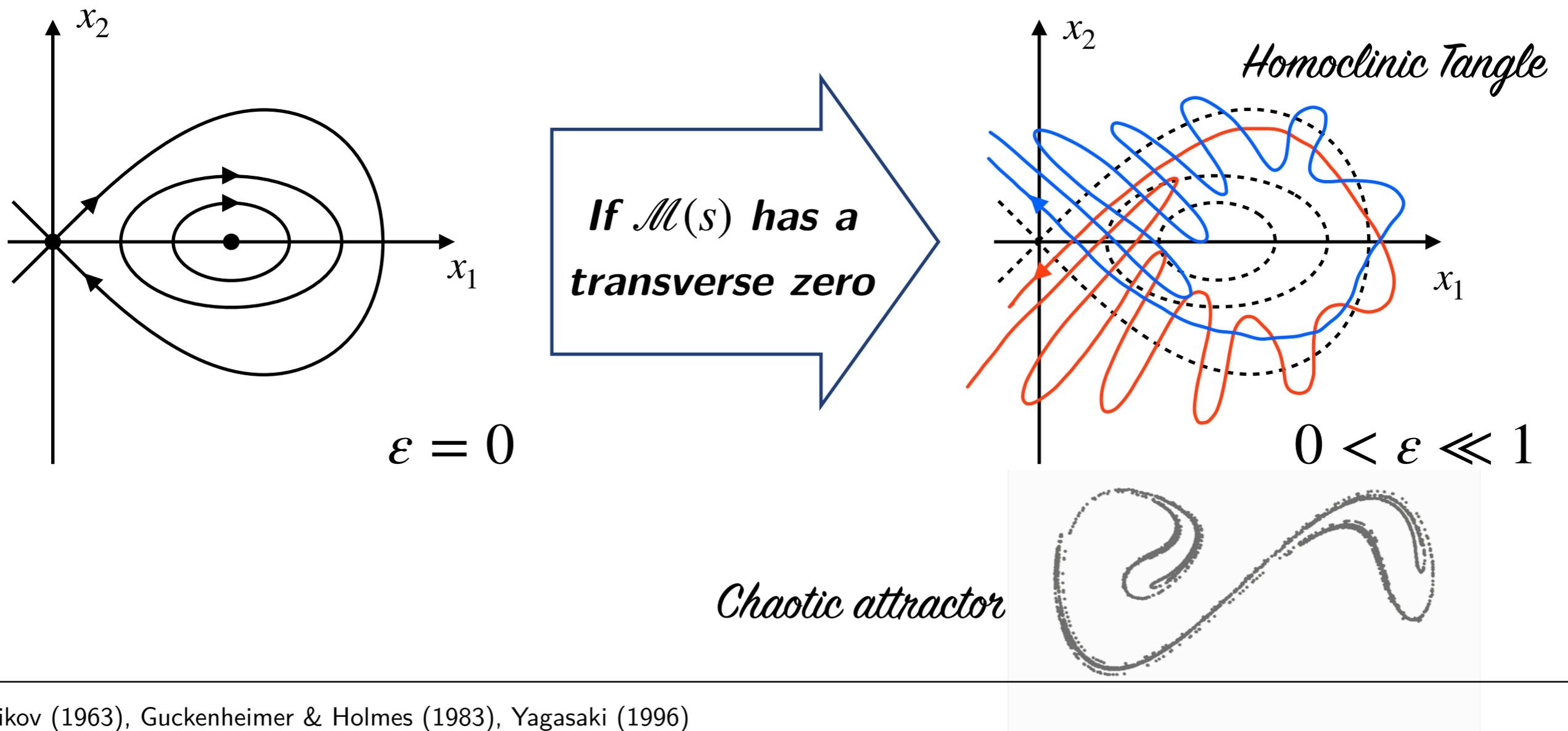
Softening

Inactive

frequency

# Overview of the Classic Melnikov Method

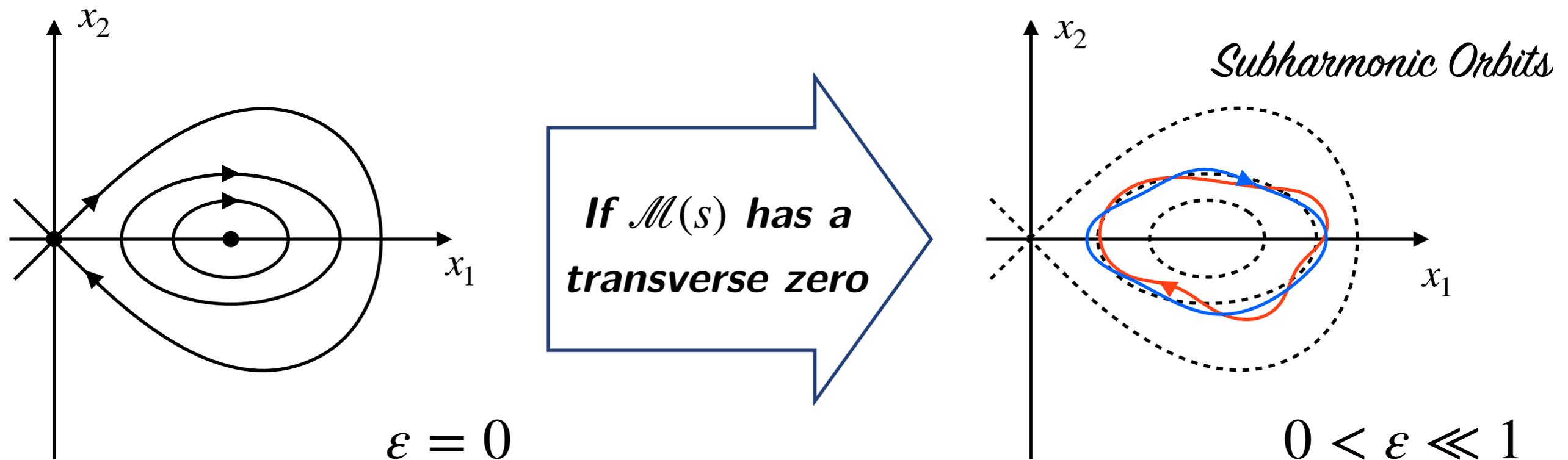
- ✦ Or better, the Poincaré-Arnold-Melnikov method (1963)
- ✦ Originally:  $\dot{x} = JDH(x) + \varepsilon g(x, t)$ ,  $g(x, t + T) = g(x, t)$ ,  $x \in \mathbb{R}^2$



# Overview of the Classic Melnikov Method

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- ✦ Extended to integrable, low-dimensional hamiltonian systems

*... not the case for structural problems in practical applications.*

# Setup: Weakly Forced and Damped Systems

- ✦ Mechanical system with  $n$  degrees of freedom, whose conservative limit is defined by the Lagrangian  $q \in \mathbb{R}^n$

$$L(q, \dot{q}) = \frac{1}{2} \langle \dot{q}, M(q) \dot{q} \rangle + \langle \dot{q}, G_1(q) \rangle + G_0(q) - V(q)$$

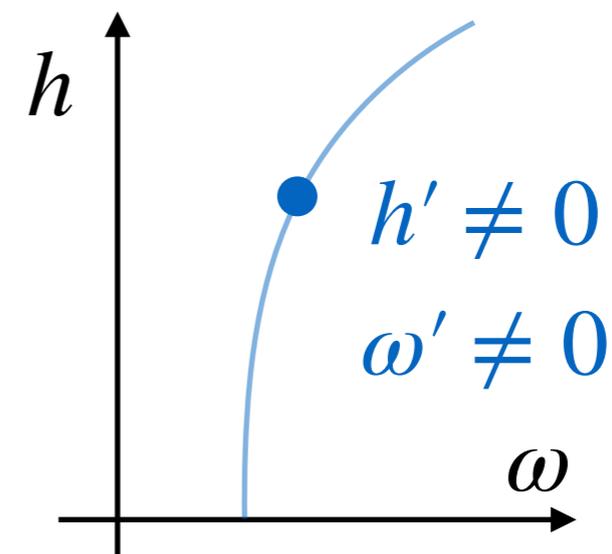
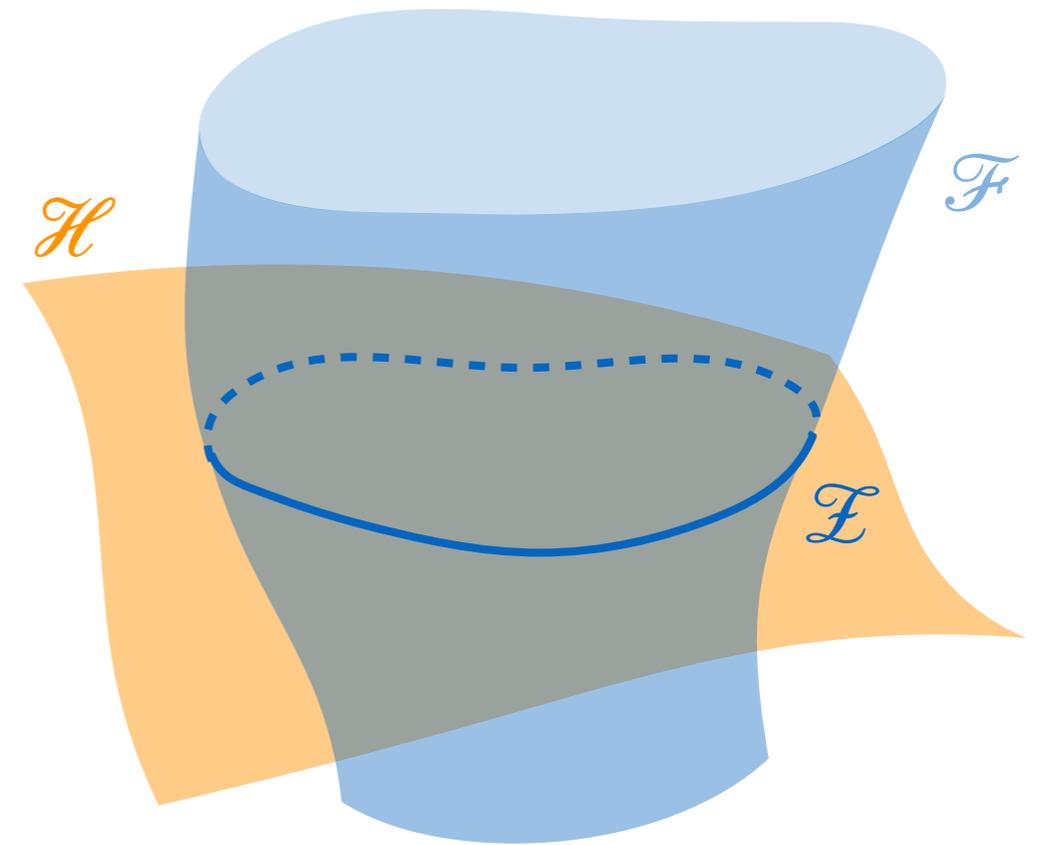
and its energy reads:  $H(q, \dot{q}) = \frac{1}{2} \langle \dot{q}, M(q) \dot{q} \rangle - G_0(q) + V(q)$

- ✦ Collecting any dissipative or active force in the small, time-periodic Lagrangian component  $Q$  with frequency  $\Omega$ , the equations of motion are

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}} \right) - \frac{\partial L}{\partial q} = \varepsilon Q(q, \dot{q}, t; \Omega, \varepsilon), \quad 0 \leq \varepsilon \ll 1$$

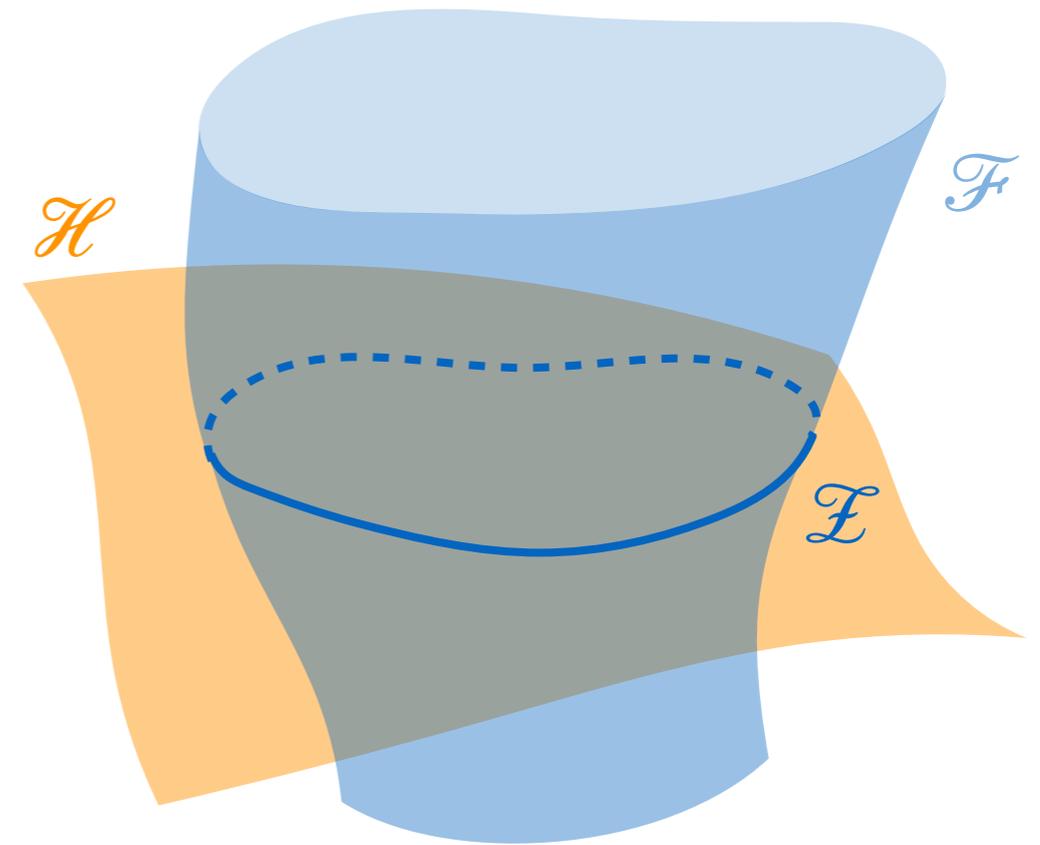
# Periodic Orbits of Conservative Systems

- ✦ Present in almost all energy levels
- ✦ Generically, they exist in families  
*NNMs*
- ✦ Not structurally stable
- ✦ Types of orbits in 1 parameter families:
  - ◆ Regular periodic orbits

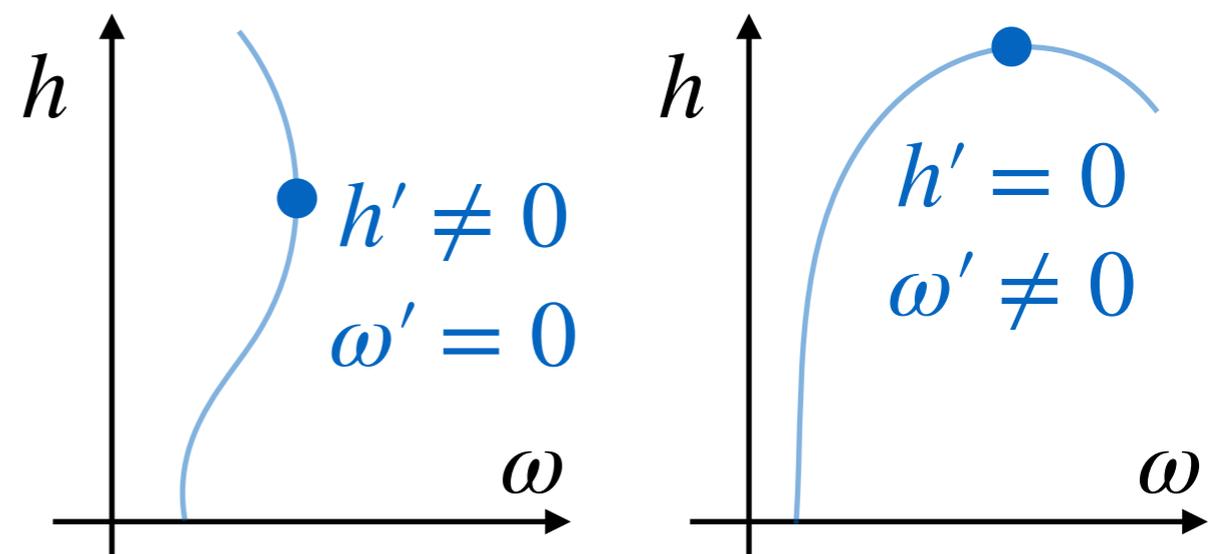


# Periodic Orbits of Conservative Systems

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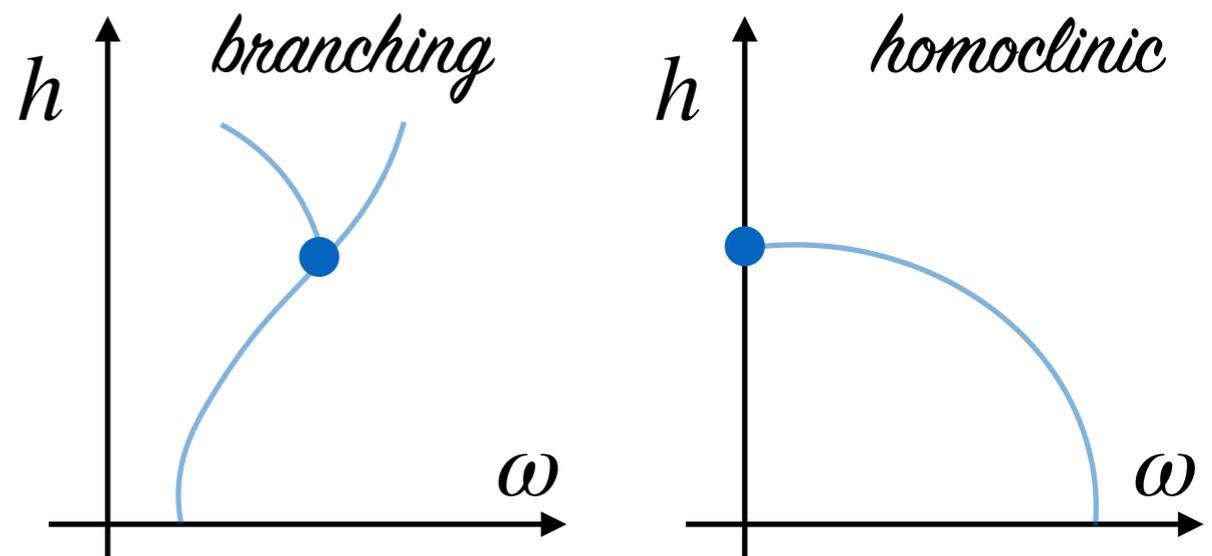
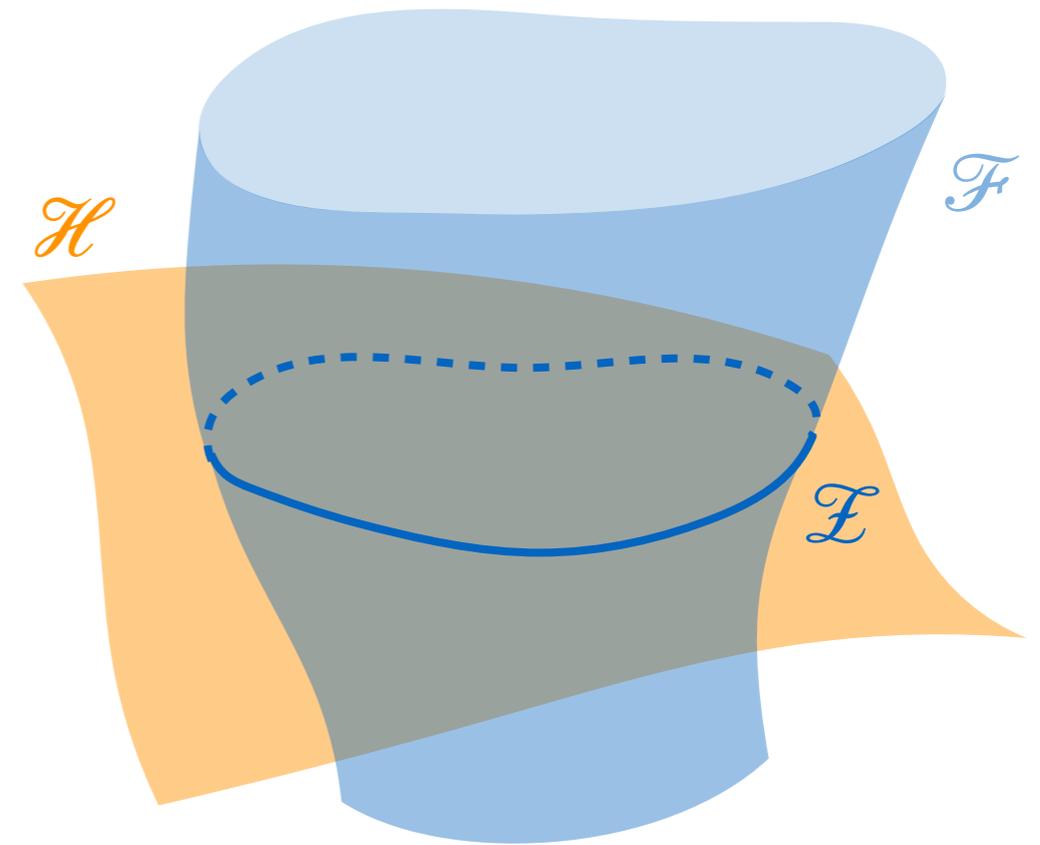


- ◆ Regular periodic orbits
- ◆ Folding periodic orbits



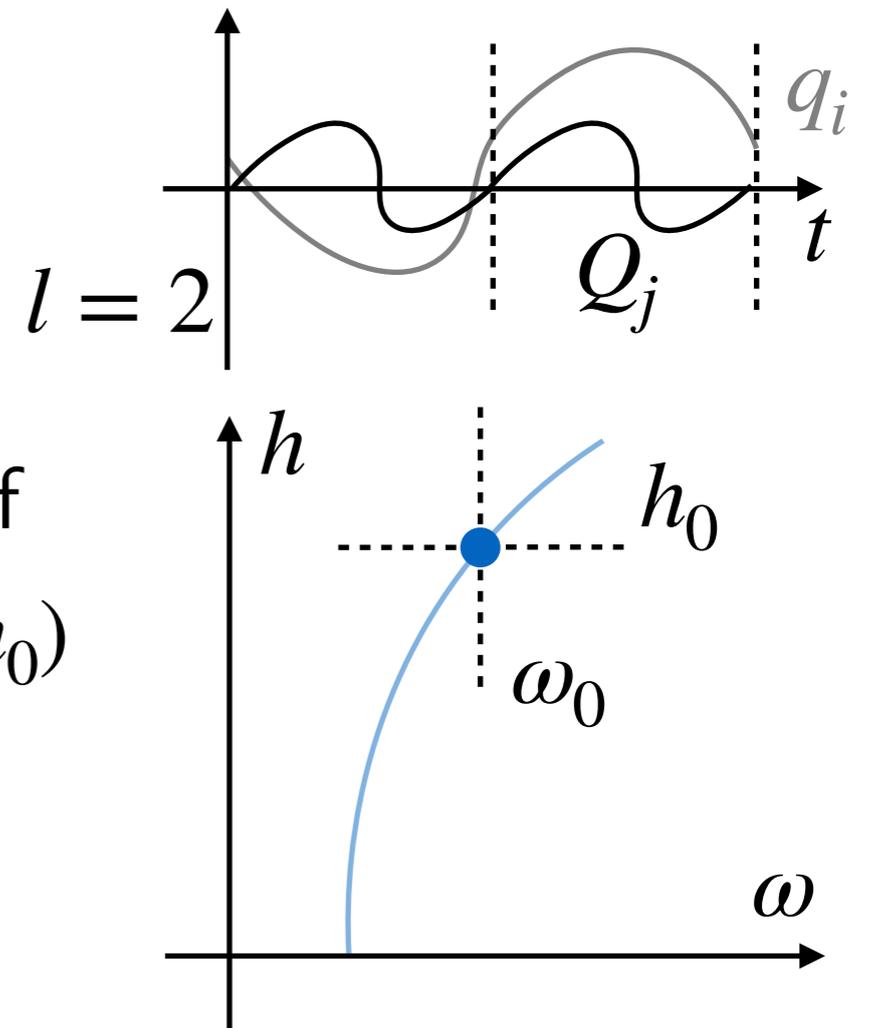
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  - ◆ Folding periodic orbits
  - ◆ ~~Critical cases~~



# Perturbation from the Conservative limit

- ✦ We look for subharmonic orbits of order  $l \in \mathbb{N}$  in the forced-damped system
- ✦ Pick a regular orbit  $q_0(t)$  with period  $\tau_0$  of the conservative backbone curve at  $(\omega_0, h_0)$
- ✦ Set  $q(t) = q_0(t + s) + O(\varepsilon)$  as well as a resonance constraint to fix  $\Omega$ , either



(a) **Exact resonance:**  $m\Omega = l\omega_0$  with  $m, l$  being relatively prime integers, or

(b) **Near resonance:**  $m\Omega = l\omega_0 + O(\varepsilon)$  and  $H(q(0), \dot{q}(0)) = h_0$

# Main Result: Existence

- Define the Melnikov function

$$\mathcal{M}_{m:l}(s) = \int_0^{m\tau_0} \langle \dot{q}_0(t+s), Q(q_0(t+s), \dot{q}_0(t+s), t; l\omega_0/m, 0) \rangle dt$$

*Work done by non-conservative forces  
evaluated at the conservative limit!*

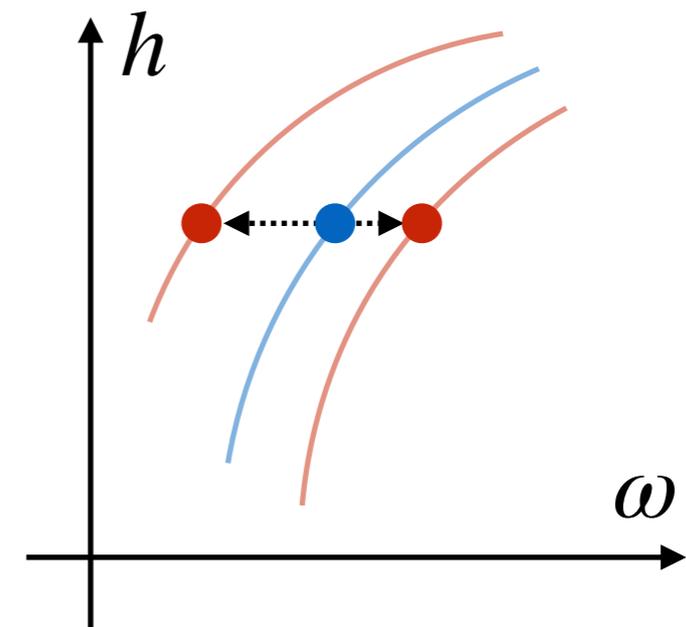
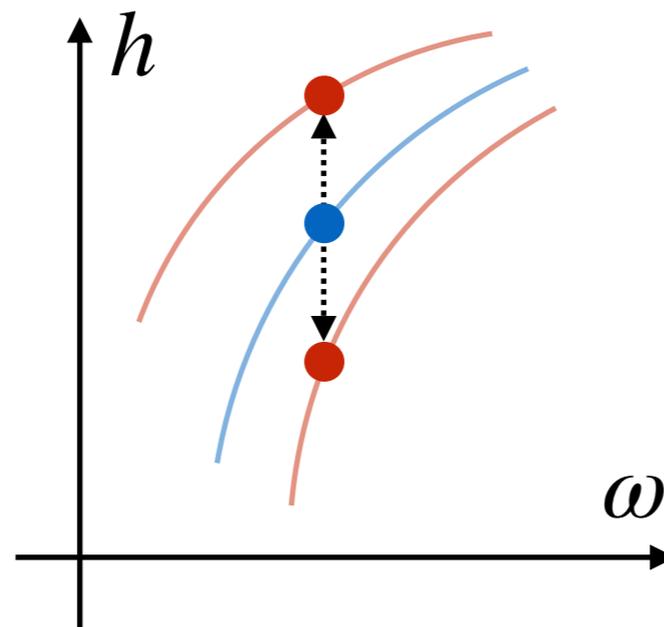
- If  $\mathcal{M}_{m:l}(s_0) = 0$  &  $\mathcal{M}'_{m:l}(s_0) \neq 0$ , the conservative limit  $q_0(s_0 + t)$  persists for the weakly damped, periodically forced system

*Exact resonance*

*Near resonance*

$\varepsilon = 0$

$0 < \varepsilon \ll 1$



# Main Result: Existence

- Define the Melnikov function

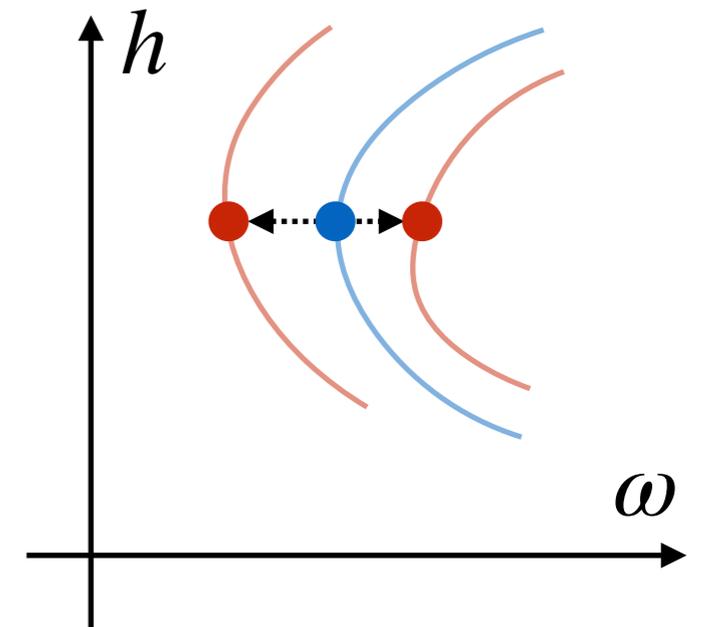
$$\mathcal{M}_{m:l}(s) = \int_0^{m\tau_0} \langle \dot{q}_0(t+s), Q(q_0(t+s), \dot{q}_0(t+s), t; l\omega_0/m, 0) \rangle dt$$

- If  $\mathcal{M}_{m:l}(s_0) = 0$  &  $\mathcal{M}'_{m:l}(s_0) \neq 0$ , but the backbone curve has a fold at  $(\omega_0, h_0)$ , then  $q_0(s_0 + t)$  persists in any direction transverse to the folding direction

*Fold in  $\omega$*

$$\varepsilon = 0$$

$$0 < \varepsilon \ll 1$$

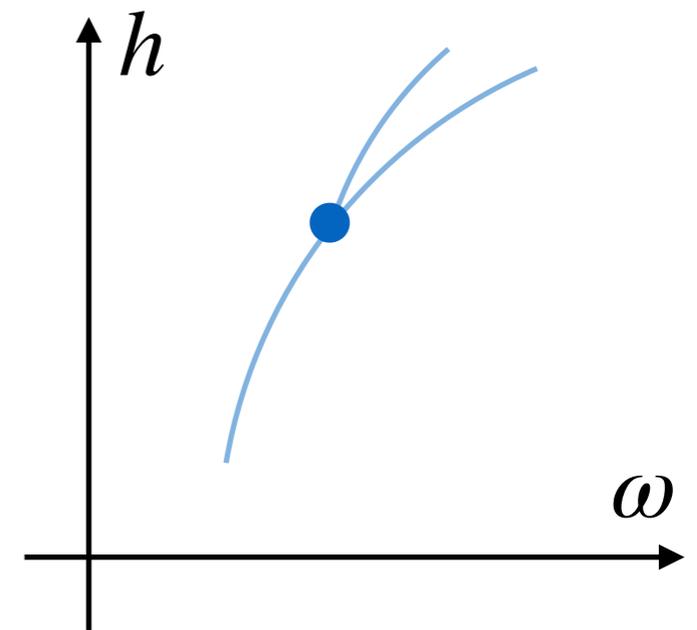


# Main Result: Existence

- ✦ Define the Melnikov function

$$\mathcal{M}_{m:l}(s) = \int_0^{m\tau_0} \langle \dot{q}_0(t+s), Q(q_0(t+s), \dot{q}_0(t+s), t; l\omega_0/m, 0) \rangle dt$$

- ✦ If  $|\mathcal{M}_{m:l}(s)| > 0$ , the conservative limit does not persist for the weakly damped, periodically forced system
- ✦ If the conservative periodic orbit  $q_0(t)$  is a critical orbit, the Melnikov function alone is not sufficient to predict the fate of the fate of  $q_0(t)$



# Towards Stability

- Write the system in Hamiltonian form

$$p = \frac{\partial L}{\partial \dot{q}} = M(q)\dot{q} + G_1(q)$$

$$x = (q, p)$$



$$J = \begin{bmatrix} 0 & I_n \\ -I_n & 0 \end{bmatrix} \quad g = \begin{pmatrix} 0 \\ Q \end{pmatrix}$$

$$\dot{x} = JDH(x) + \varepsilon g(x, t; \Omega, \varepsilon)$$

- For the stability of a periodic orbit with period  $l\Omega$  we need to study the eigenvalues of the monodromy matrix  $X(l\Omega) \in \mathbb{R}^{n \times n}$

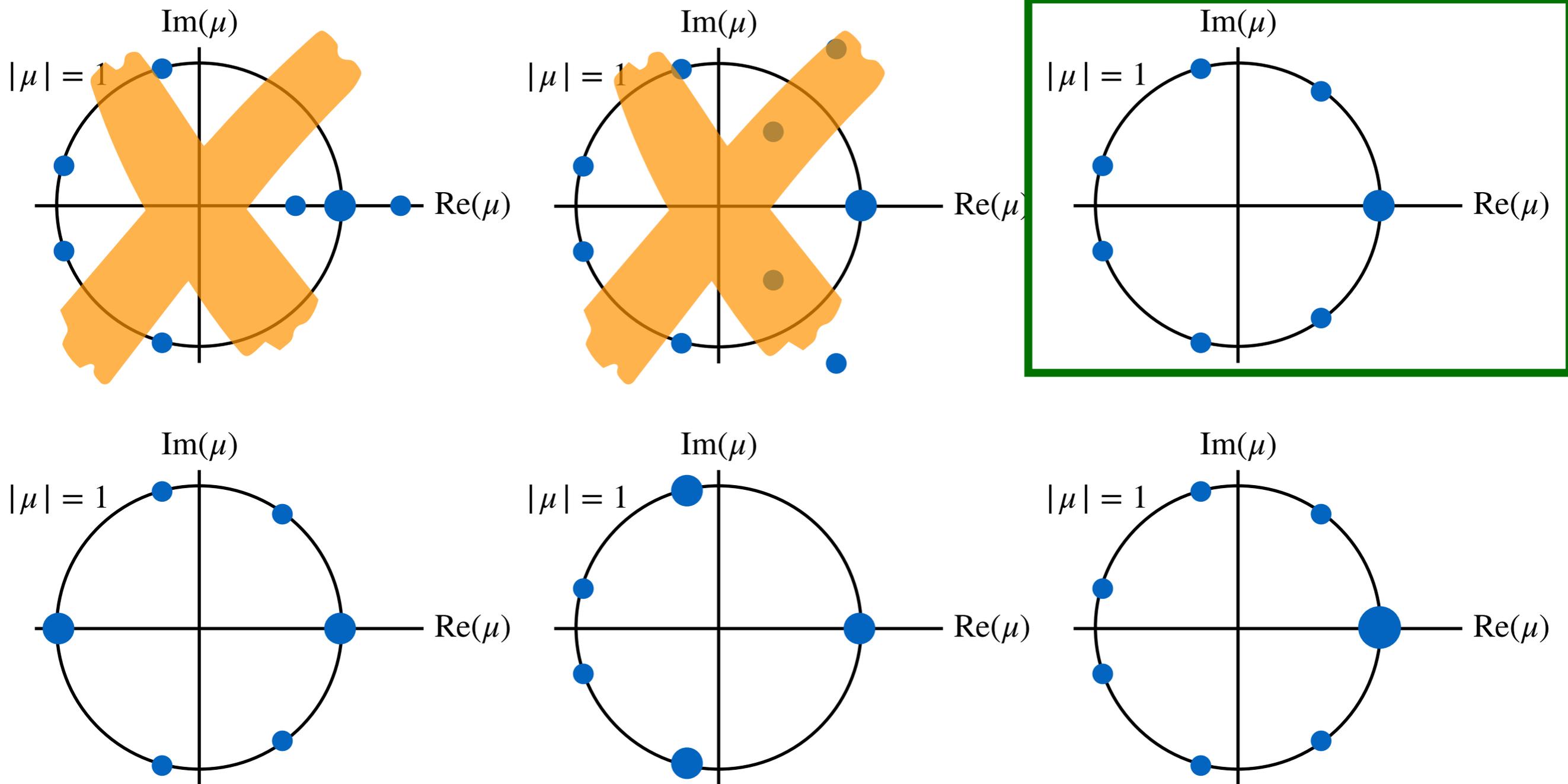
$$\dot{X} = JD^2H(x(t))X + \varepsilon D_x g(x(t), t; \Omega, \varepsilon)$$

$$X(0) = I$$

$\Pi_0 = X_0(m\tau_0)$  is the solution at  $\varepsilon = 0$

# Towards Stability

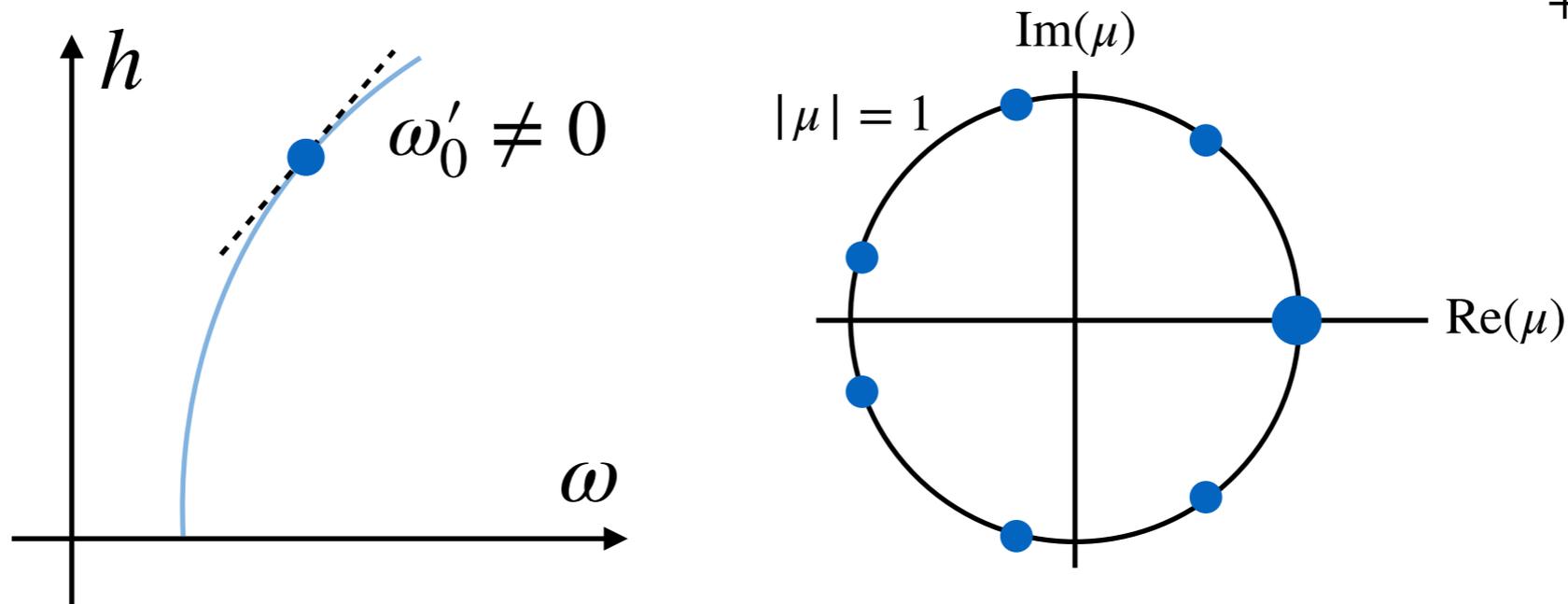
The conservative limit has always at least 2 eigenvalues of  $\Pi_0$  equal to  $+1$ . Possible configurations of the unperturbed spectrum



# Towards Stability

- ✦ We consider a conservative limit that satisfies

*Note: the eigenvalue +1 is not regular*



- ✦ For each of the  $n$  couples of eigenvalues, define the **nonlinear damping rate**

$$C_i = -\frac{1}{m\tau_0} \int_0^{m\tau_0} \text{trace} \left( S_i X_0^{-1}(t) D_x g(x_0(t), t; l\omega_0/m, 0) X_0(t) R_i \right) dt$$

$\text{span}(R_i)$  is the  $i$ -th eigenspace,  $S_i = (R_i^\top J R_i)^{-1} R_i^\top J$  and  $\dot{X}_0 = J D^2 H(x_0(t)) X_0$ ,  $X_0(0) = I$

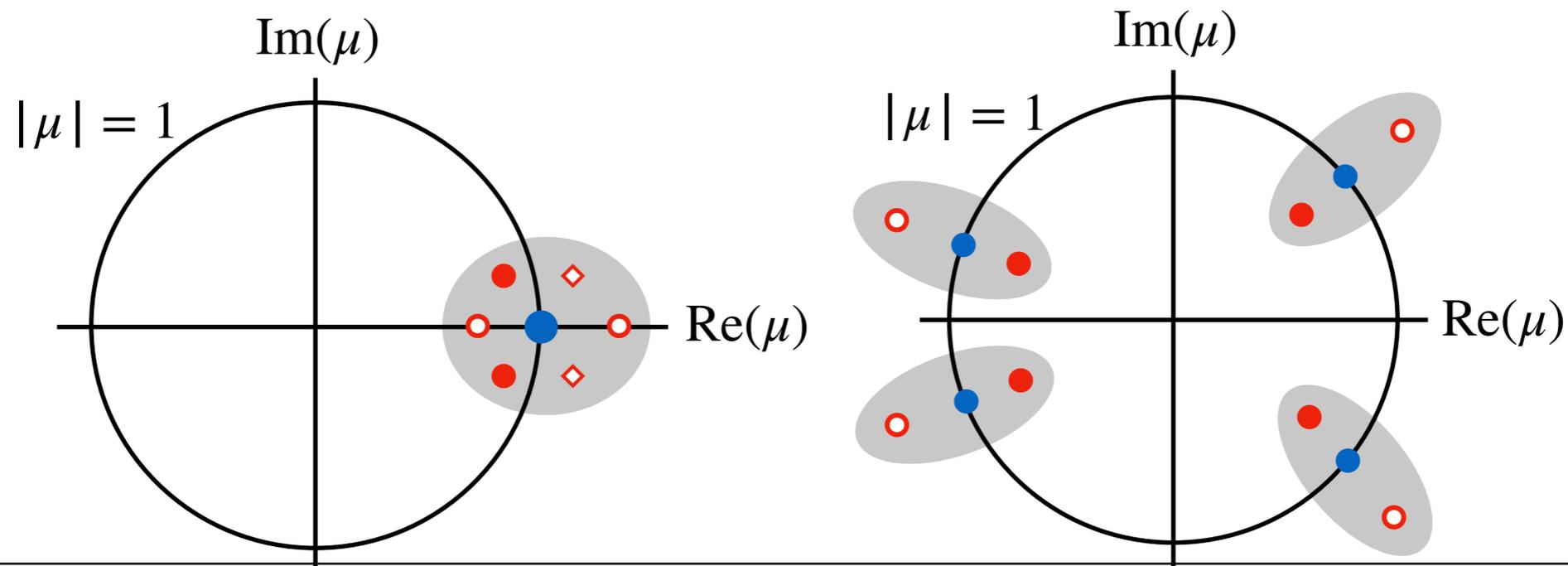
# Main Result: Stability

The forced-damped periodic orbit is **unstable** if

$$\mathcal{M}'_{m:l}(s_0)\omega'_0 < 0 \quad \underline{\text{or}} \quad \exists i \in \{1, \dots, n\} : C_i < 0$$

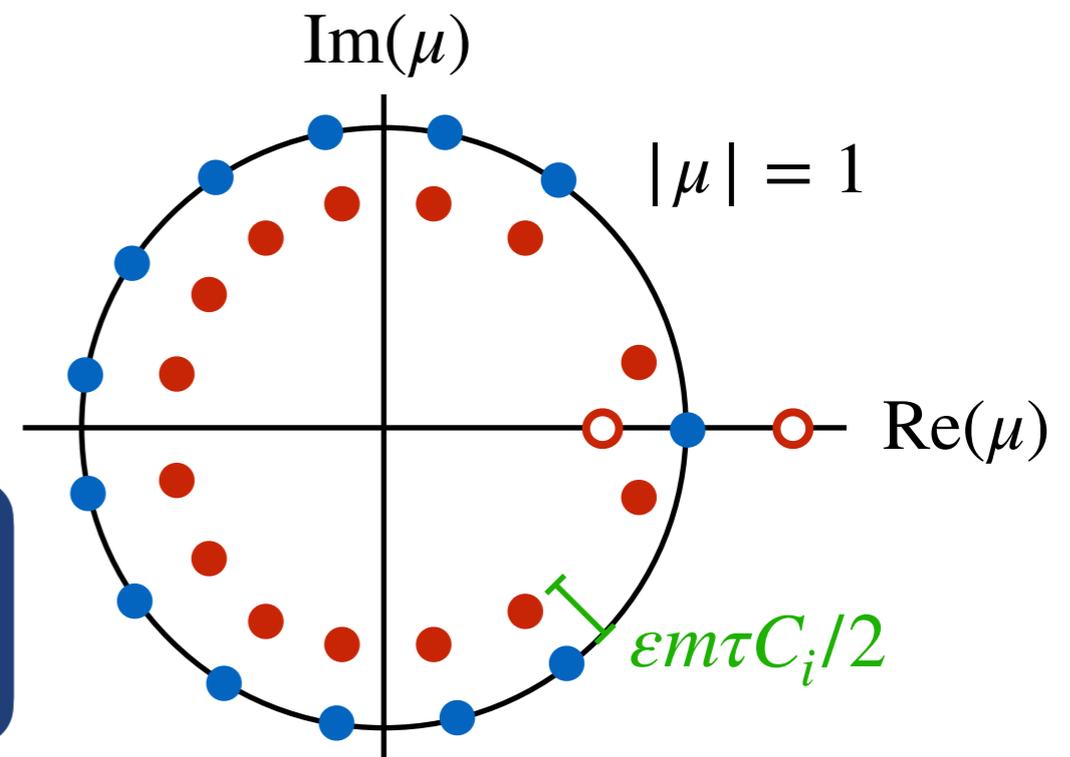
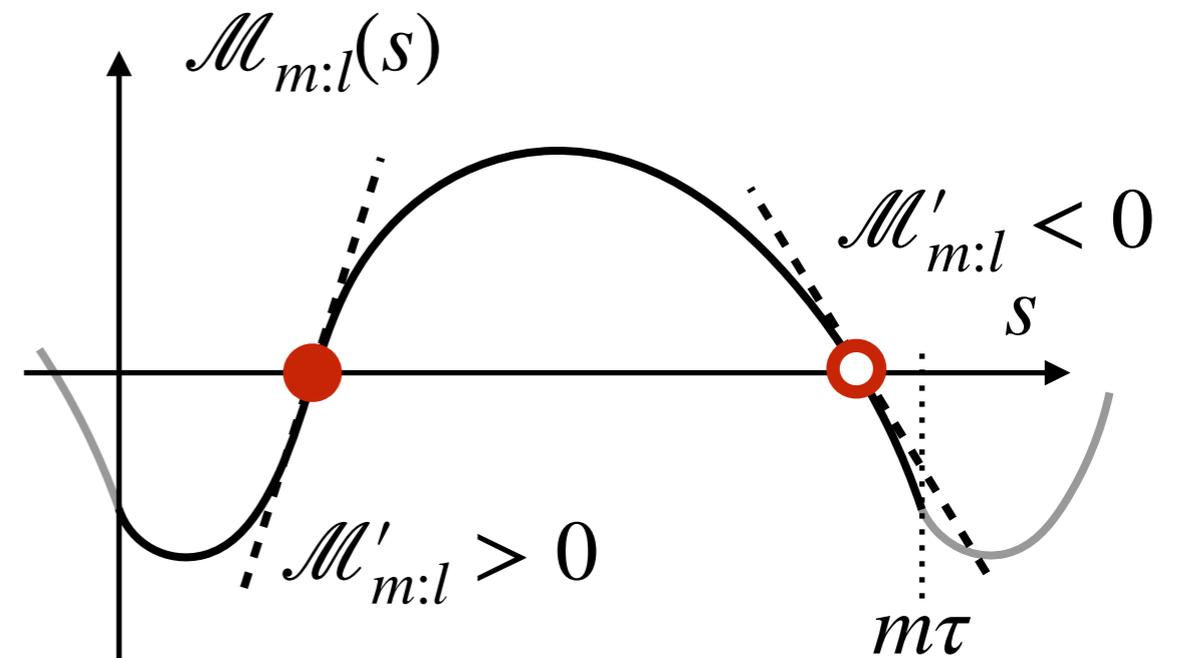
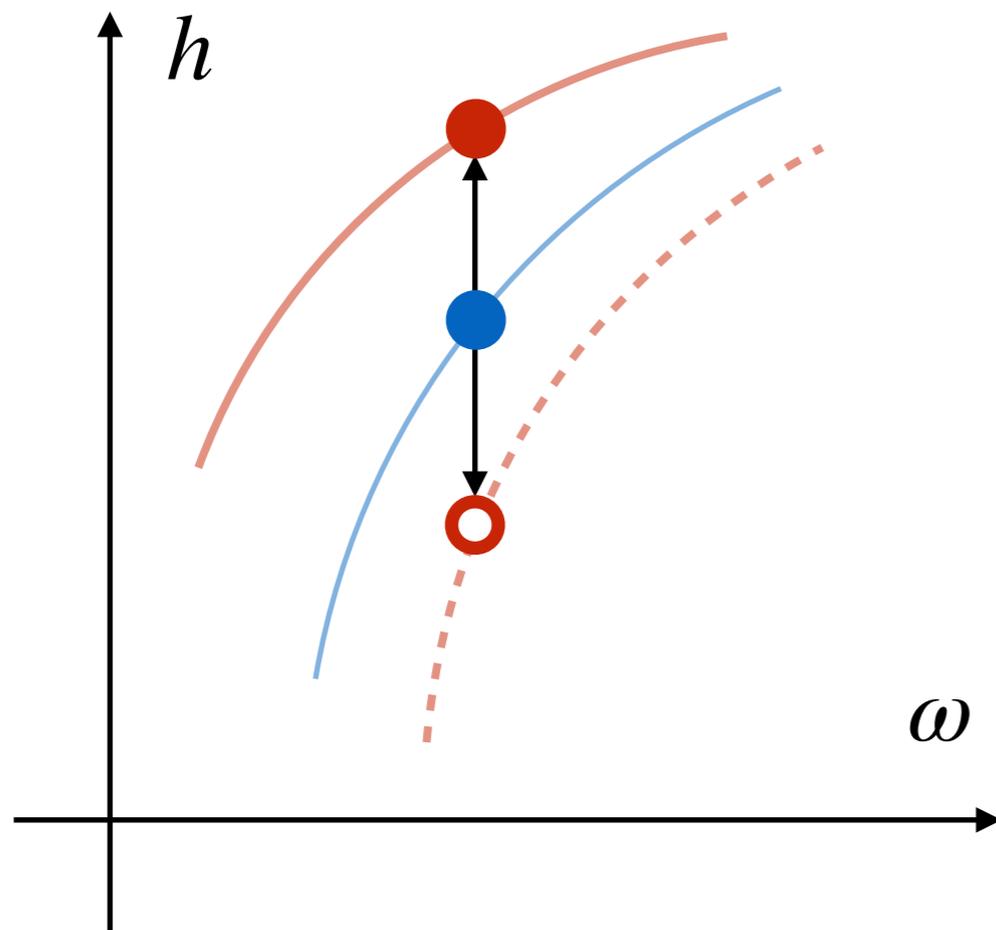
The forced-damped periodic orbit is **asymptotically stable** if

$$\mathcal{M}'_{m:l}(s_0)\omega'_0 > 0 \quad \underline{\text{and}} \quad C_i > 0 \quad \forall i \in \{1, \dots, n\}$$



# Connection with Experimental Observations

- Assume that the nonlinear damping rates are positive



*These predictions are obtained without any simulation of the forced-damped system*

# Remarks

- ✦ The formula for the nonlinear damping rate is complex.

$$C_i = -\frac{1}{m\tau_0} \int_0^{m\tau_0} \text{trace} \left( S_i X_0^{-1}(t) D_x g(x_0(t), t; l\omega_0/m, 0) X_0(t) R_i \right) dt$$

- ✦ For  $n = 1$ ,  $C_1 = -\frac{1}{m\tau_0} \int_0^{m\tau_0} \text{trace} \left( D_x g(x_0(t), t; l\omega_0/m, 0) \right) dt$

- ✦ For  $Q = F(t) - \alpha M(q)p$ , then  $C_i = \alpha \forall i \in \{1, \dots, n\}$

- ✦ Instability conditions can be formulated for other cases

# Example: Subharmonics in a Gyro

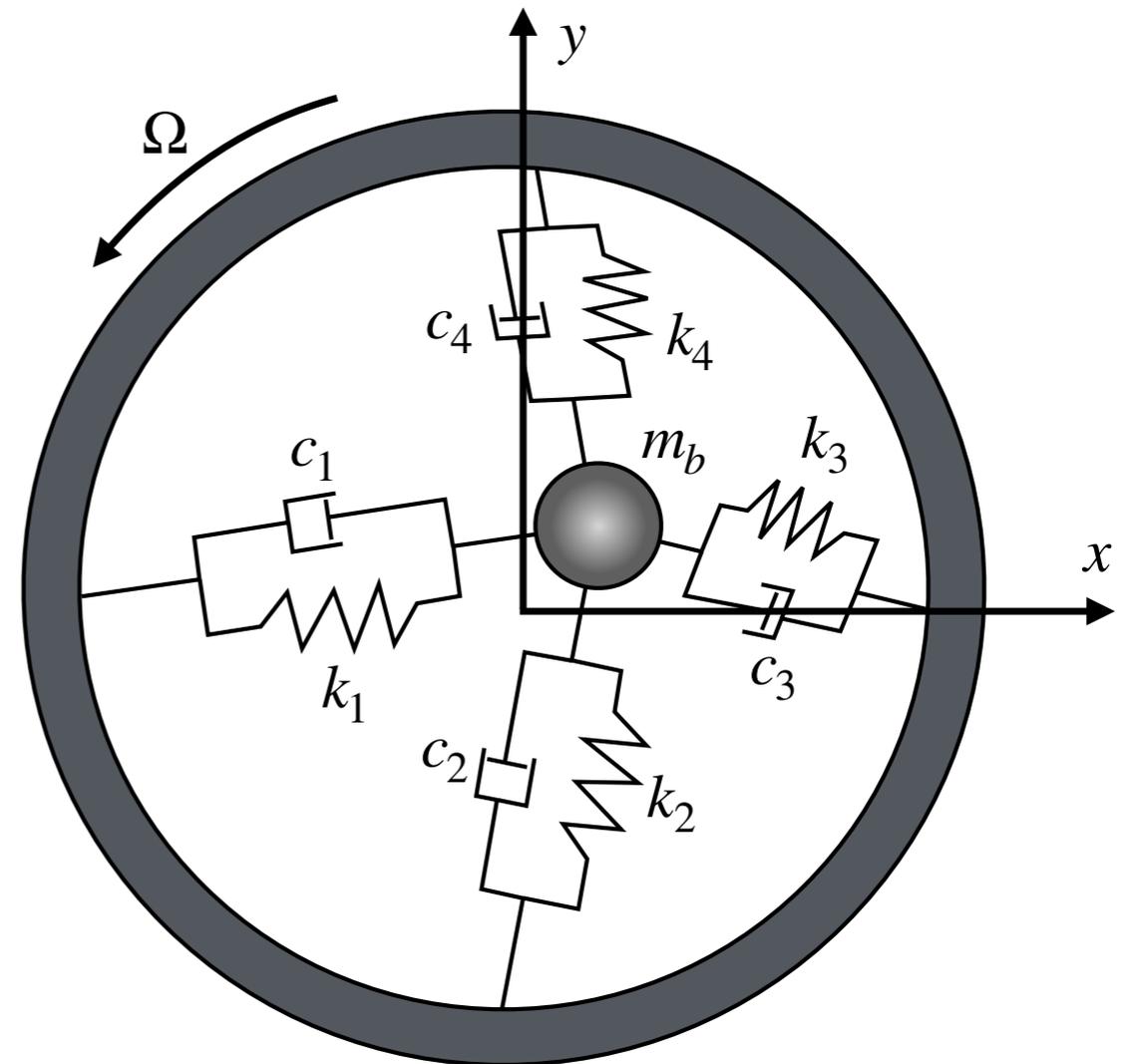
$$m_b \ddot{q} + 2G\dot{q} - m_b \Omega^2 q + DV(q) = \hat{Q}(q, \dot{q}, t)$$

$$G = m_b \begin{bmatrix} 0 & -\Omega \\ \Omega & 0 \end{bmatrix},$$

$$V(q) = \frac{1}{2} \sum_{j=1}^4 k_j (l_j(x, y) - l_0)^2,$$

$$l_{1,3}(x, y) = \sqrt{(l_0 \pm x)^2 + y^2},$$

$$l_{2,4}(x, y) = \sqrt{x^2 + (l_0 \pm y)^2},$$



$$\Omega = 0.942, l_0 = 1, k_1 = 1, k_2 = 4.08, k_3 = 1.37, k_4 = 2.51$$

# Example: Subharmonics in a Gyro

$$m_b \ddot{q} + 2G\dot{q} - m_b \Omega^2 q + DV(q) = \hat{Q}(q, \dot{q}, t)$$

$$\hat{Q}(q, \dot{q}, t) = \varepsilon (Q_{d,\alpha}(q, \dot{q}) + Q_{d,\beta}(q, \dot{q}) + Q_f(t))$$

- ✦ Damping linearly depending on the absolute velocities the mass  $m_b$  (e.g. air damping)  $\varepsilon Q_{d,\alpha}(q, \dot{q}) = -\varepsilon \alpha m_b (\dot{q} + m_b^{-1} Gq)$ ;
- ✦ Stiffness-proportional damping for the spring-damper elements, i.e.  $c_j = \varepsilon \beta k_j$  for  $j = 1, \dots, 4$  and  $\varepsilon Q_{d,\beta}(q, \dot{q}) = -\varepsilon \beta C(q) \dot{q}$ ,

$$C(q) = \sum_{j=1}^4 k_j \begin{bmatrix} (\partial_x l_j(x, y))^2 & \partial_x l_j(x, y) \partial_y l_j(x, y) \\ \partial_x l_j(x, y) \partial_y l_j(x, y) & (\partial_y l_j(x, y))^2 \end{bmatrix}$$

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- ✦ Mono-harmonic forcing of frequency  $l\Omega$

$$\varepsilon Q_f(t) = \varepsilon \begin{pmatrix} +\cos(l\Omega t) \\ -\sin(l\Omega t) \end{pmatrix}, \quad l \in \mathbb{N}.$$

# Example: Subharmonics in a Gyro

- ✦ Equations of motion in Hamiltonian form

$$\dot{q} = -Gq + p,$$

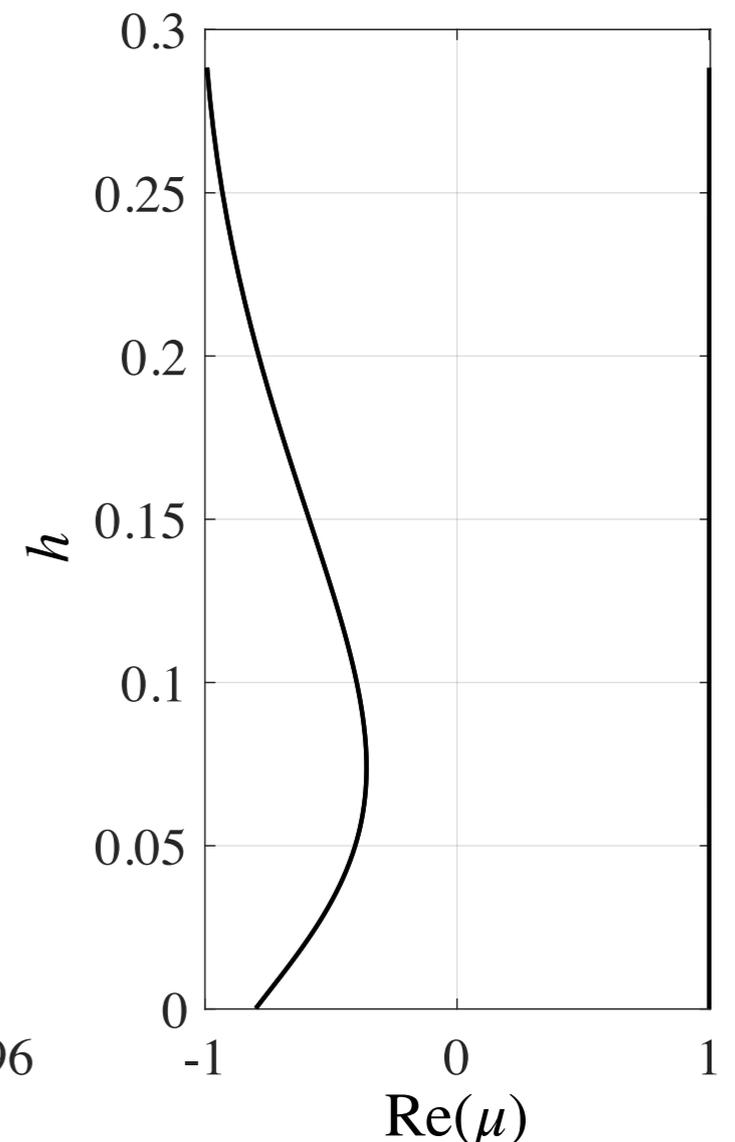
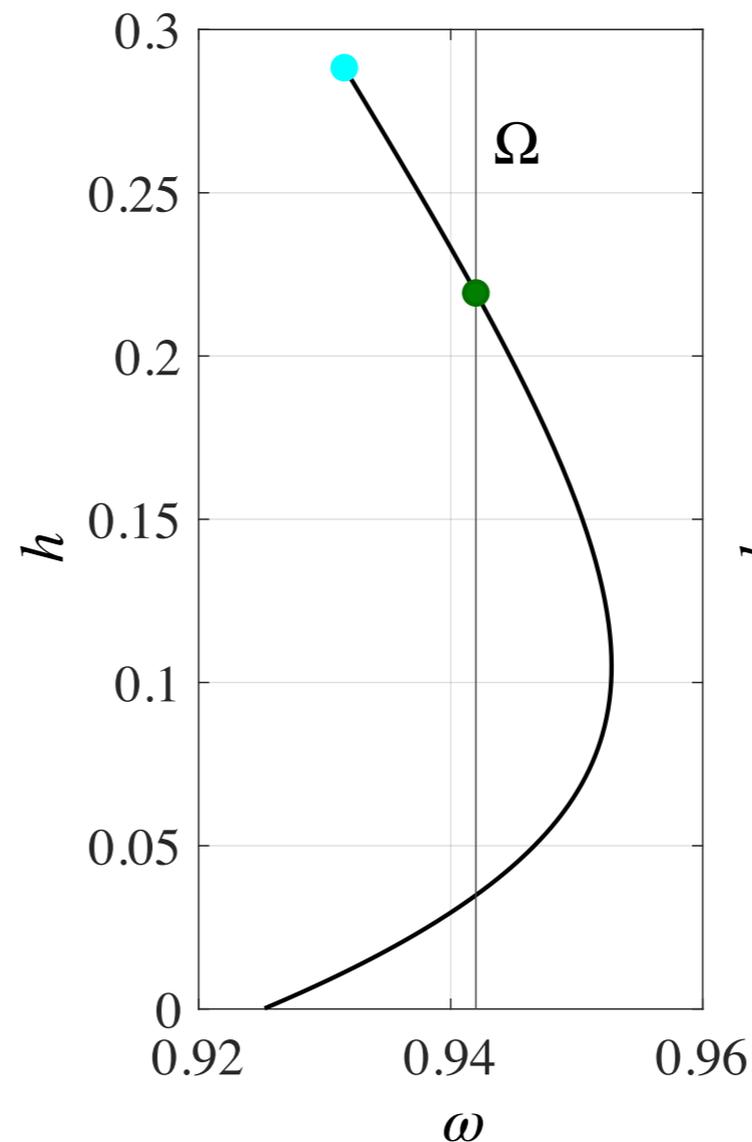
$$\dot{p} = -DV(q) - Gp + \varepsilon(Q_f(t) - \alpha p - \beta C(q)(p - Gq)).$$

- ✦ Conservative limit:

- ◆ Linearized frequencies at the equilibrium

0.92513 and 3.1431

- ◆ Focus on the first NNM



# Example: Subharmonics in a Gyro

- ✦ Equations of motion in Hamiltonian form

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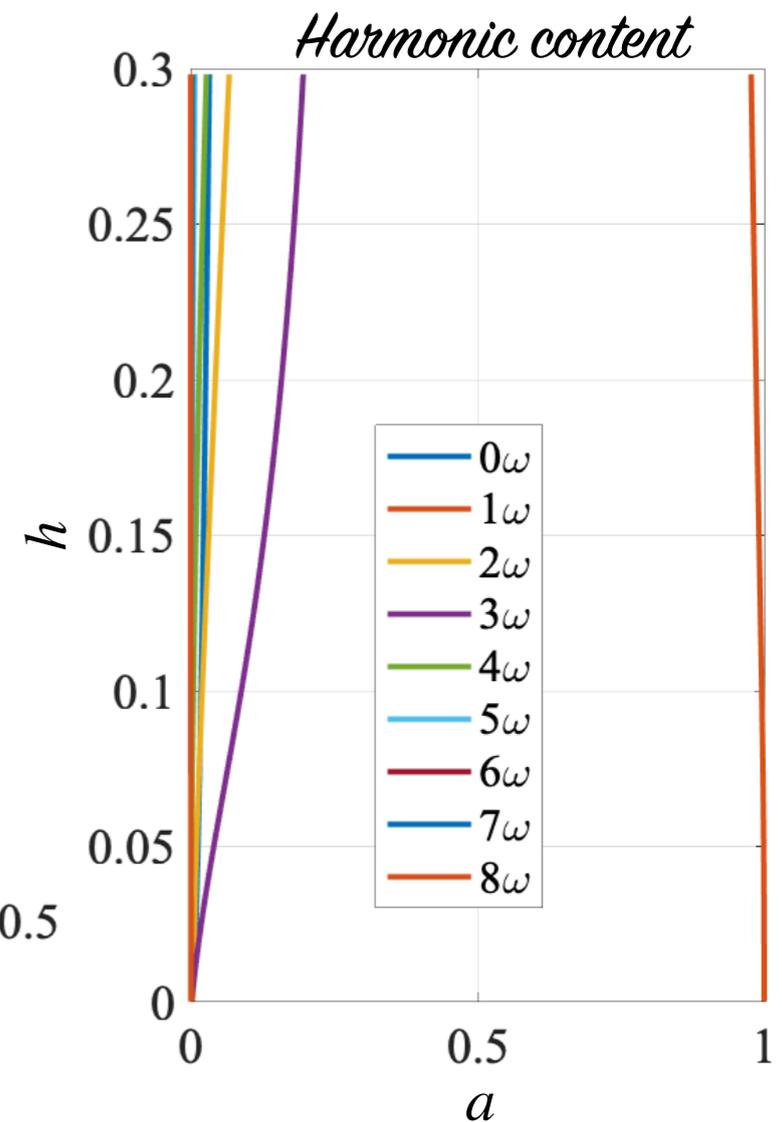
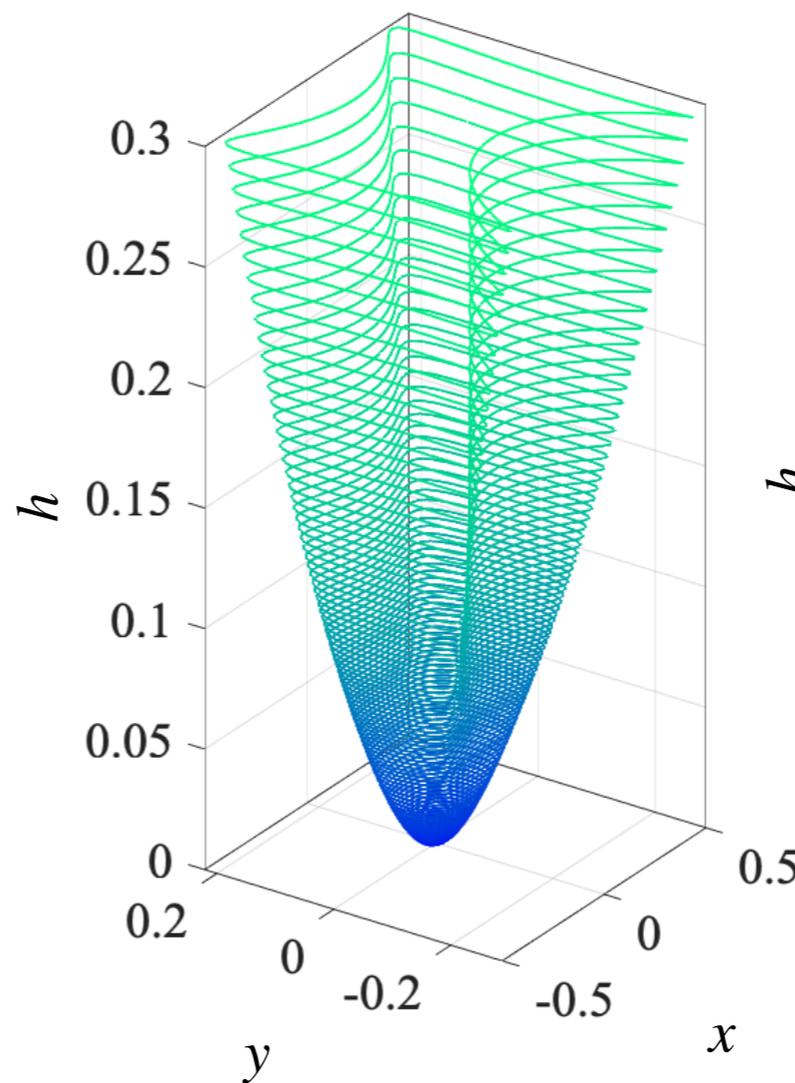
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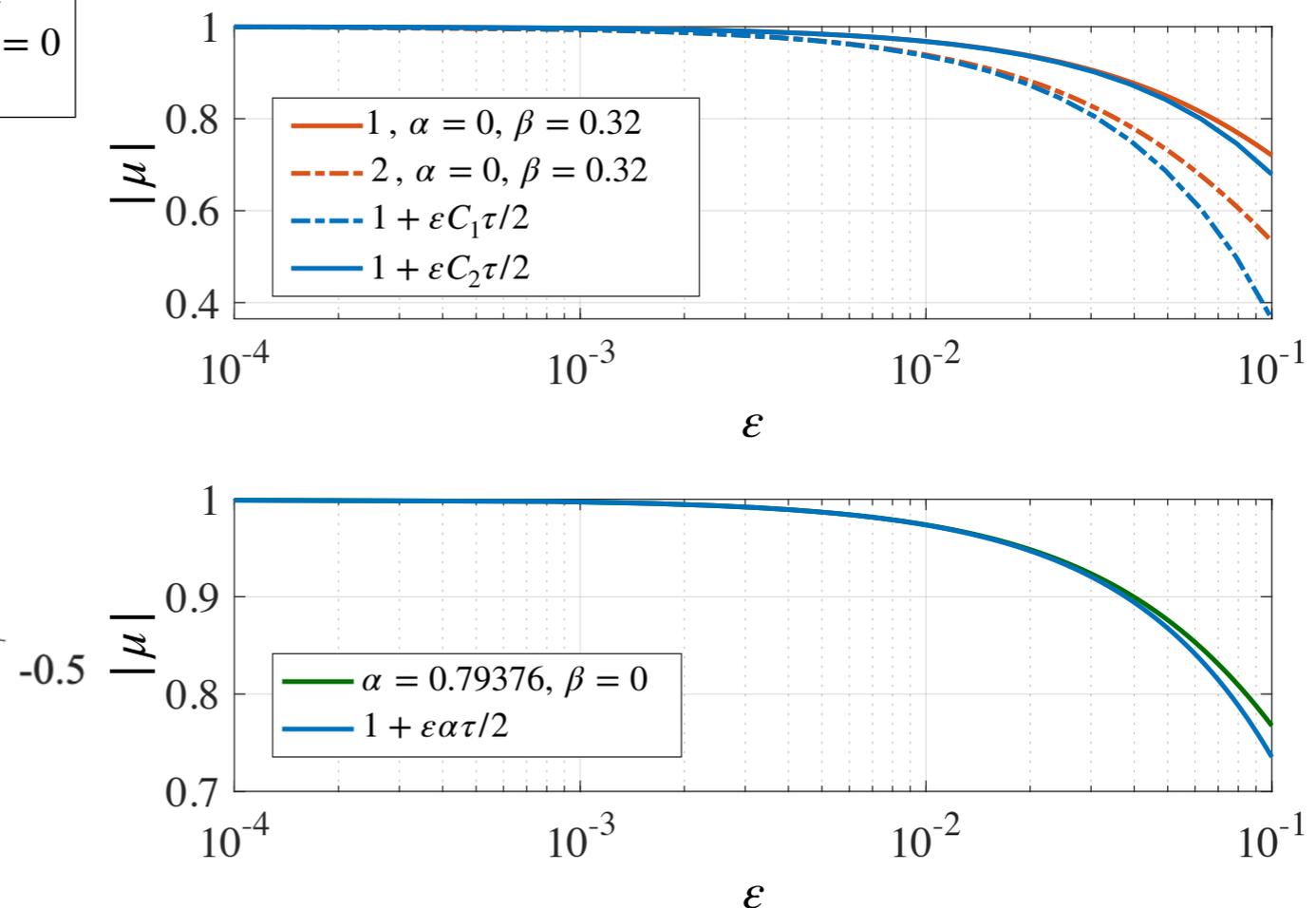
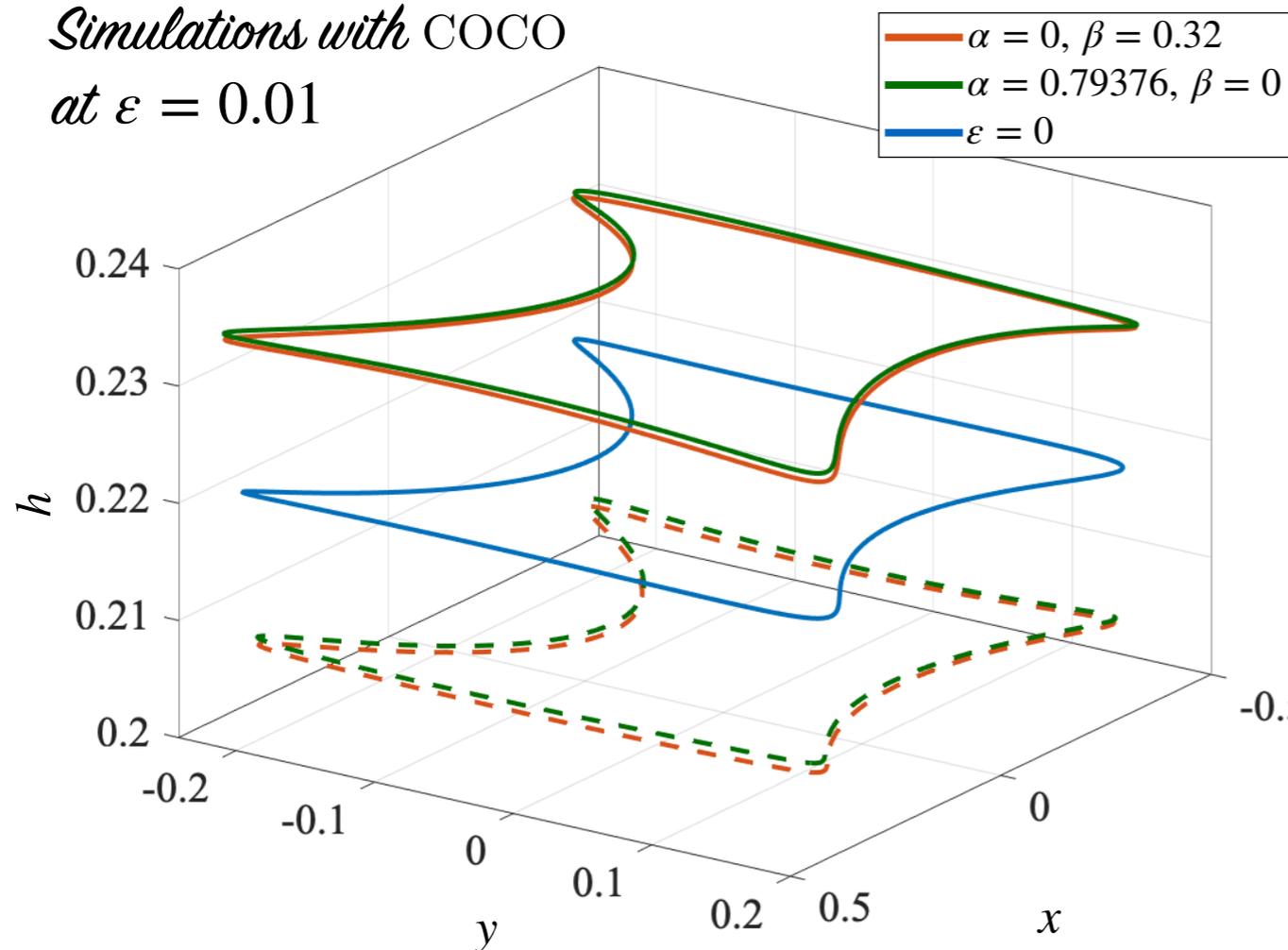
- ◆ Set  $l = 3$



# Example: Subharmonics in a Gyro

- Analysis of the two separate damping mechanisms
- The Melnikov function is  $\mathcal{M}_{1:3}(s) = 1.4402 \cos(3\Omega s) - 1.1553$  for  $\alpha = 0.76376, \beta = 0$  and  $\alpha = 0, \beta = 0.32$

*Simulations with COCO*  
at  $\varepsilon = 0.01$



# From Single Orbits to Orbits Families

- ✦ We have a framework to study eventual **singular behaviors** when varying a parameter  $\kappa$

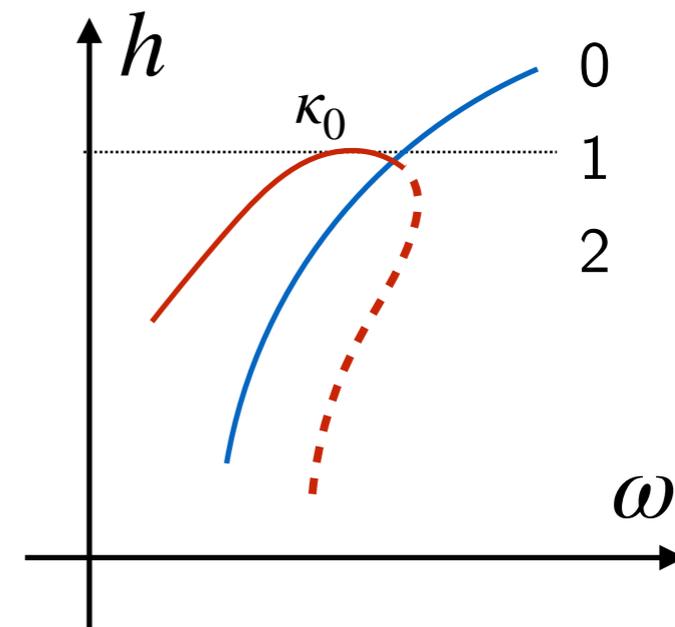
- ✦ We focus on quadratic zeros, defined as:

$$\mathcal{M}_{m:l}(s_0, \kappa_0) = D_s \mathcal{M}_{m:l}(s_0, \kappa_0) = 0 \quad D_{ss} \mathcal{M}_{m:l}(s_0, \kappa_0) \neq 0$$

- ✦ The simplest case is the one of **limit point** (codim. 0)

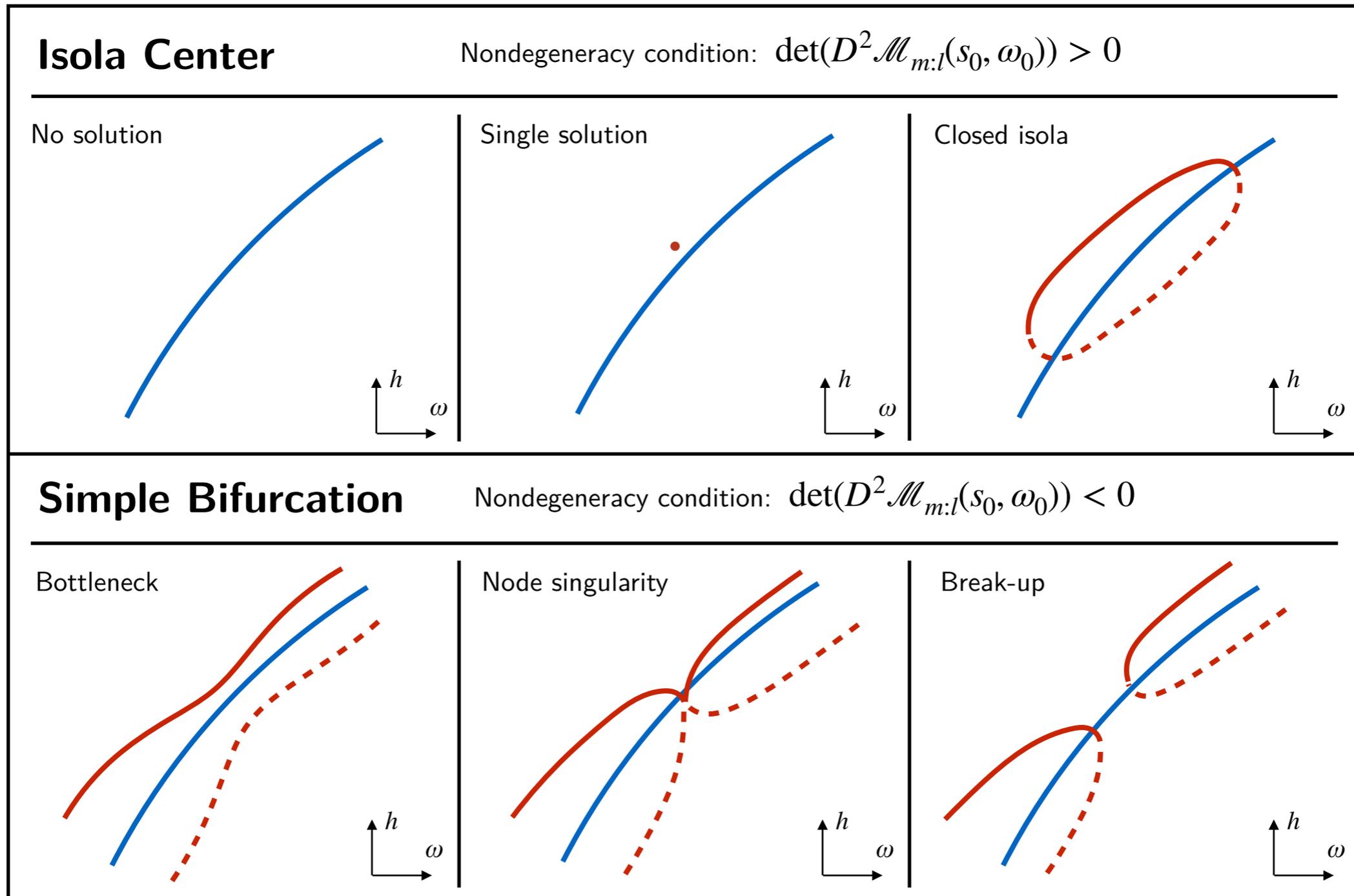
$$D_\kappa \mathcal{M}_{m:l}(s_0, \kappa_0) \neq 0$$

Detection of maximal responses



# A Zoo of Bifurcations

Defining conditions:  $\mathcal{M}_{m:l}(s_0, \omega_0) = D_s \mathcal{M}_{m:l}(s_0, \omega_0) = D_\omega \mathcal{M}_{m:l}(s_0, \omega_0) = 0$



# Example: Parametric Forcing and Isolating

$$\dot{q} = p,$$

$$q, p \in \mathbb{R}^3$$

$$\dot{p}_1 = -k(q_1 - q_2) - k/3q_1 - aq_1^2 - bq_1^3 - \varepsilon\alpha p_1,$$

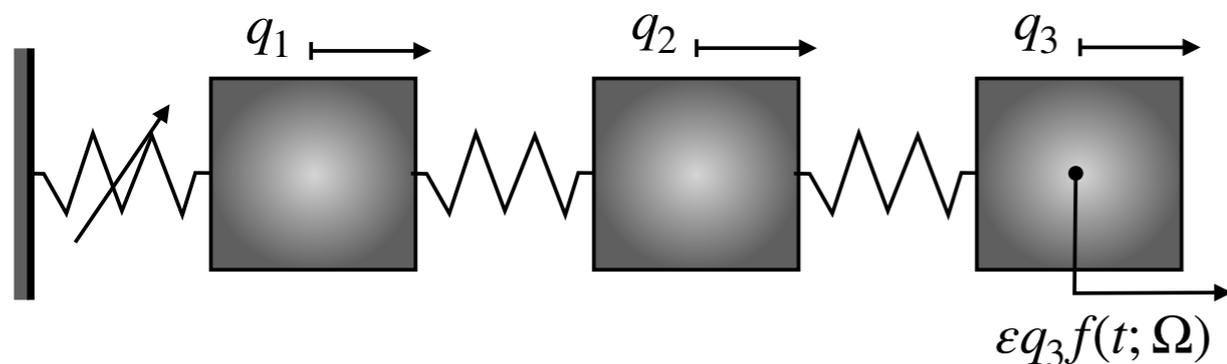
$$k = 1,$$

$$\dot{p}_2 = -k(q_2 - q_1) - k(q_2 - q_3) - \varepsilon\alpha p_2,$$

$$a = -0.5,$$

$$\dot{p}_3 = -k(q_3 - q_2) + \varepsilon(q_3 f(t; \Omega) - \alpha p_3),$$

$$b = 1,$$

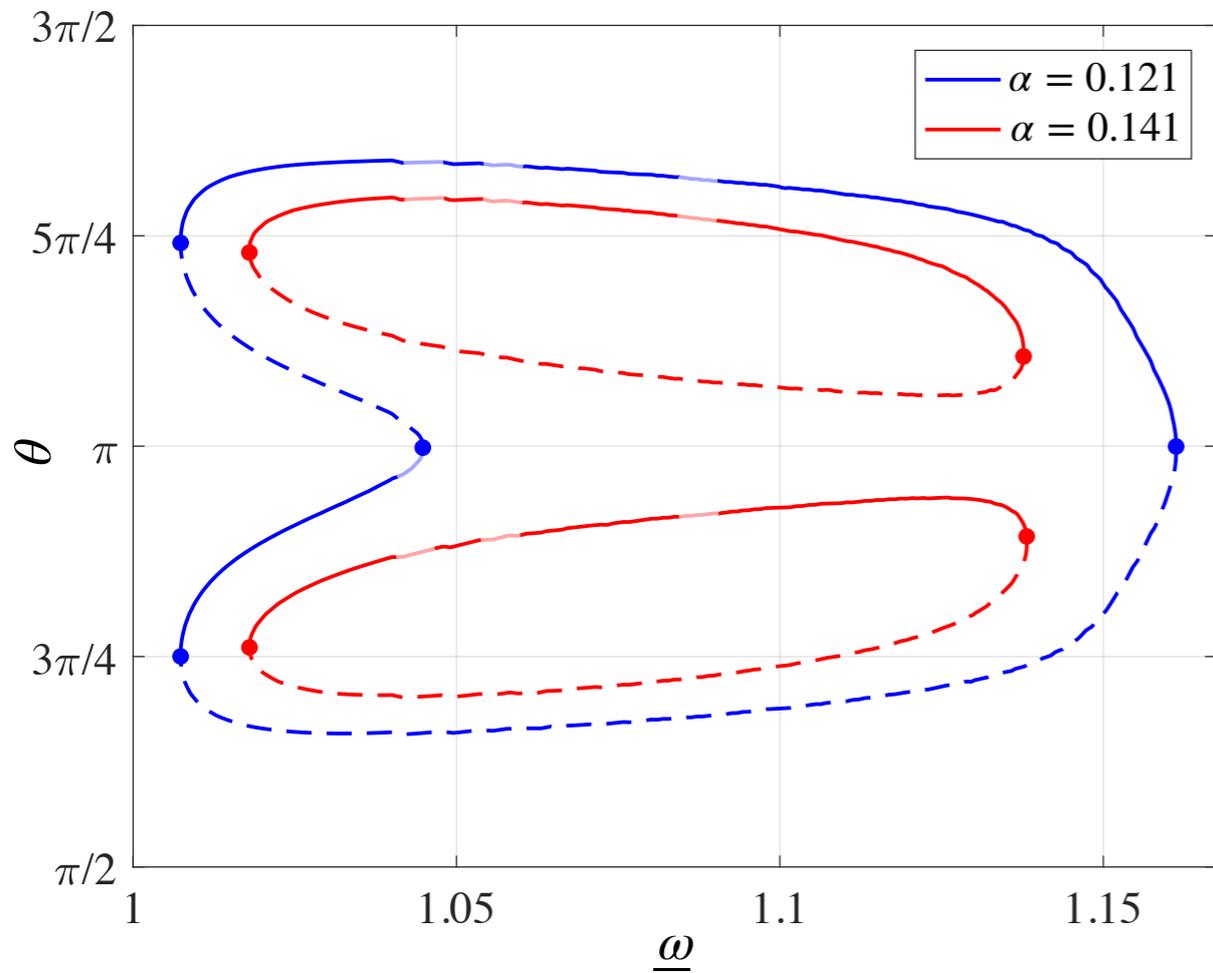


*Approximation of a square-wave up to the 6-th harmonic*

$$f(t; \Omega) = \frac{4}{\pi} \sum_{j=1}^3 \frac{1}{2j-1} \sin((2j-1)\Omega t),$$

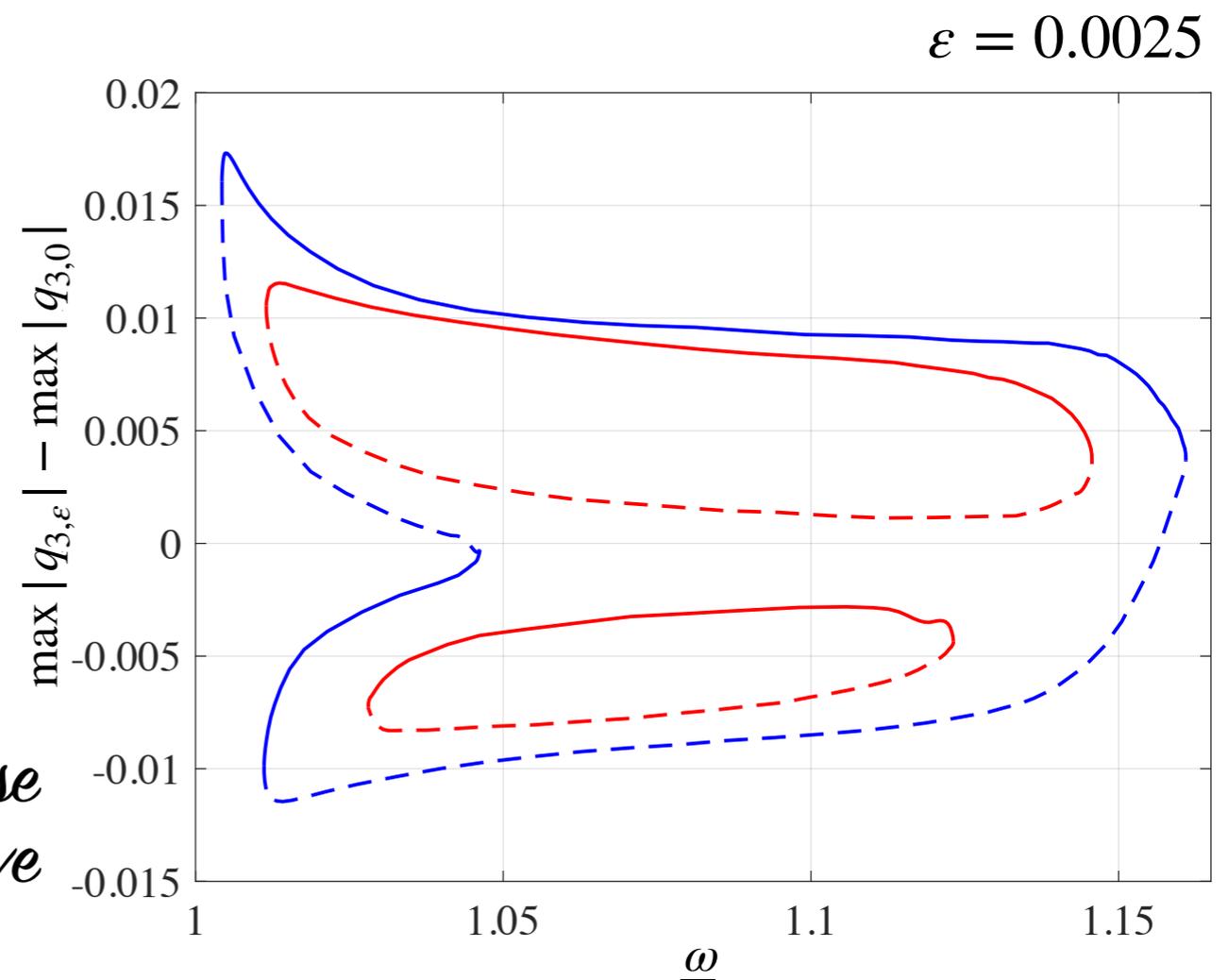
# Example: Parametric Forcing and Isolals

- Assume a 1:1 resonance and evaluate  $\mathcal{M}_{1:1}$  along the family



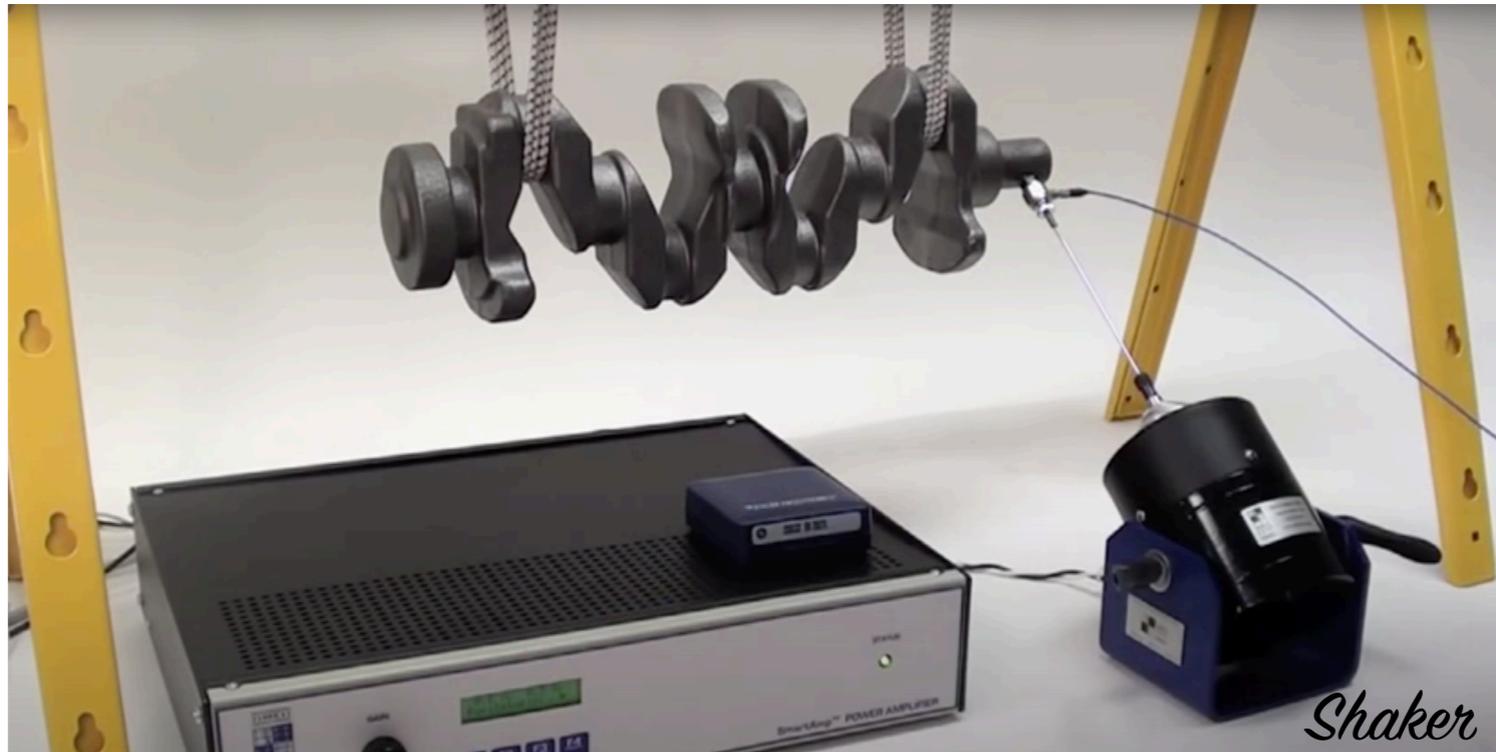
*Loci of zeros of the Melnikov function as function of the frequency and the phase shift of the orbit*

*Distance of the frequency response from the backbone curve*



# Experimental Applications

Testing for backbone curve extraction

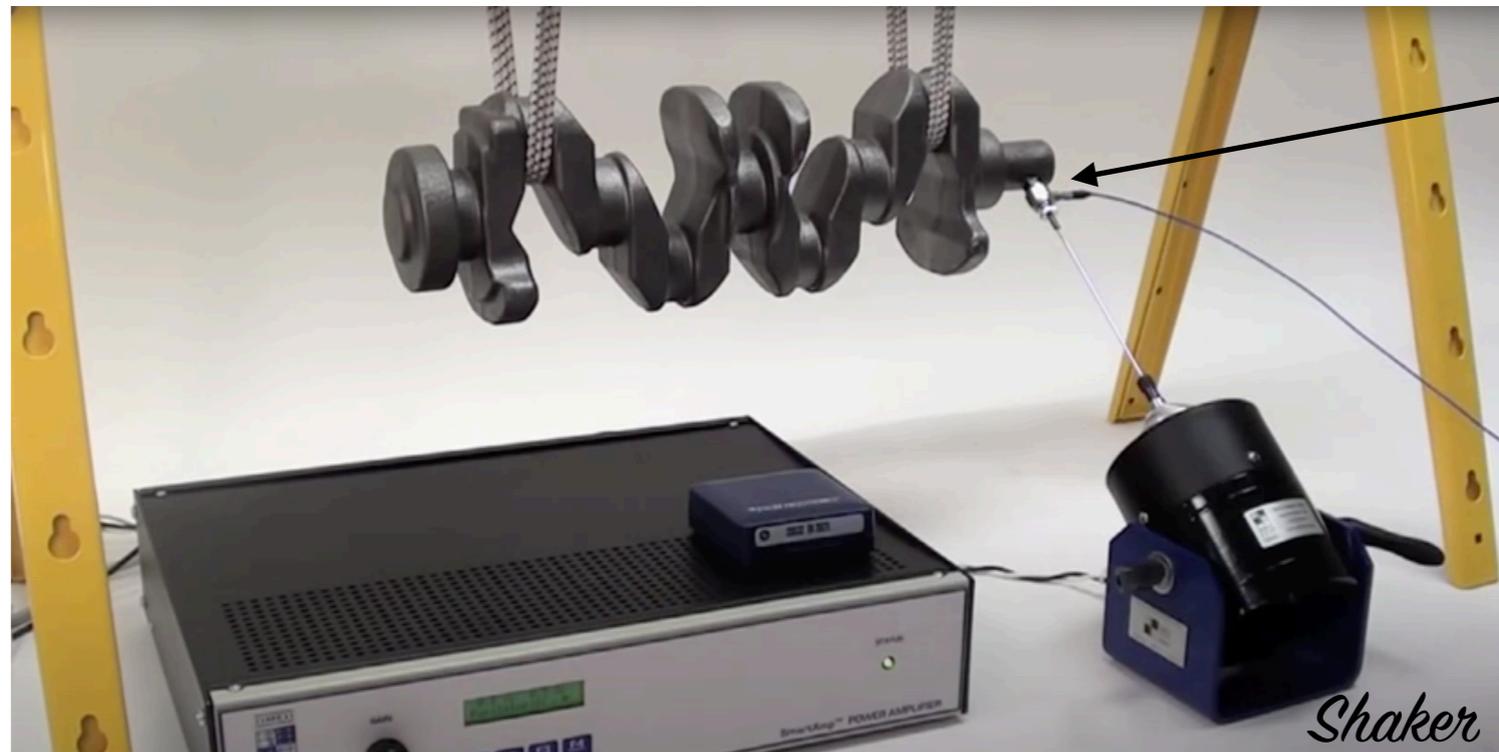


**Phase-lag quadrature criterion:** forcing is exactly balancing the damping if the phase lag between forcing and measurement is  $90^\circ$

This was show for: synchronous motions and linear damping

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Testing for backbone curve extraction



*Co-located  
accelerometer*

**Phase-lag quadrature criterion:** forcing is exactly balancing the damping if the phase lag between forcing and measurement is  $90^\circ$

*Using our Melnikov analysis one can show that this is valid, when forcing is mono-harmonic, for arbitrary motions and damping shapes, but only for co-located measurements!*

# Summary and Future Directions

- ✦ An energy balance is sufficient to establish the existence of weakly forced-damped vibrations from the conservative limit, while their stability can be studied with nonlinear damping rates
- ✦ These analytical results matches with available ones for single-degree-of-freedom oscillators and with real life observations
- ✦ Our approach offers significant advantages both for numerical and experimental studies
- ✦ What about the survival of tori?

